



# **Transitionsmodellierung technischer Strömungen**

**Florian Menter; Robin Langtry  
ANSYS Germany, 83624 Otterfing  
[Florian.Menter@ansys.com](mailto:Florian.Menter@ansys.com)**

# Transition Modelling in Industrial CFD



## Effects

- Re number effects
- Heat transfer
- Wall shear stress
- Separation behaviour
- Efficiency of many technical devices

## Modelling

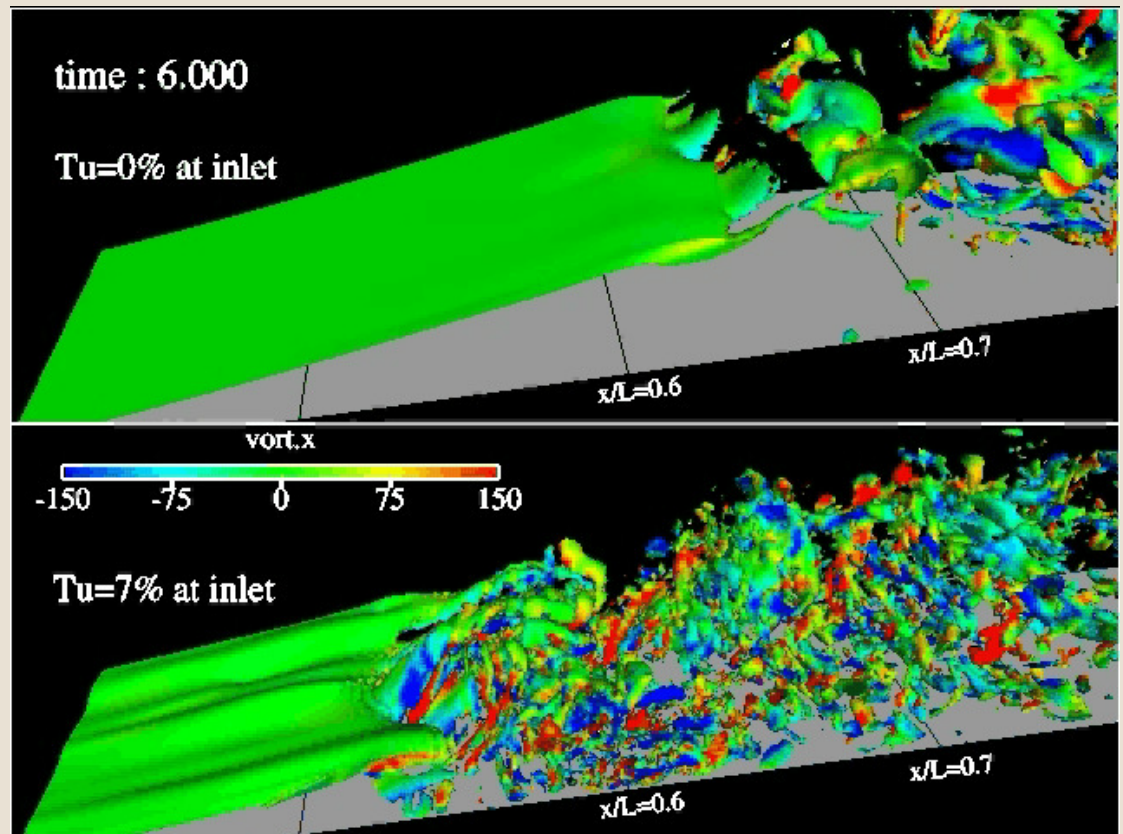
- Numerous developments:
  - Correlation based models
  - Low-Re models
  - $e^n$  linear stability
  - PSE models
  - LES
  - DNS

**~75% of all technical flows are in a Re range of  $10^4$ - $10^6$  and therefore in transitional regime**

**Almost all industrial CFD simulations are calculated without a transition model**

## Spanwise vorticity iso-surfaces (Sims. 1,2)

- Wakes impact on boundary layer cause bypass transition
- $Re=60,000$
- Periodic in spanwise direction
- No of grid nodes  $\sim 30 \times 10^6$



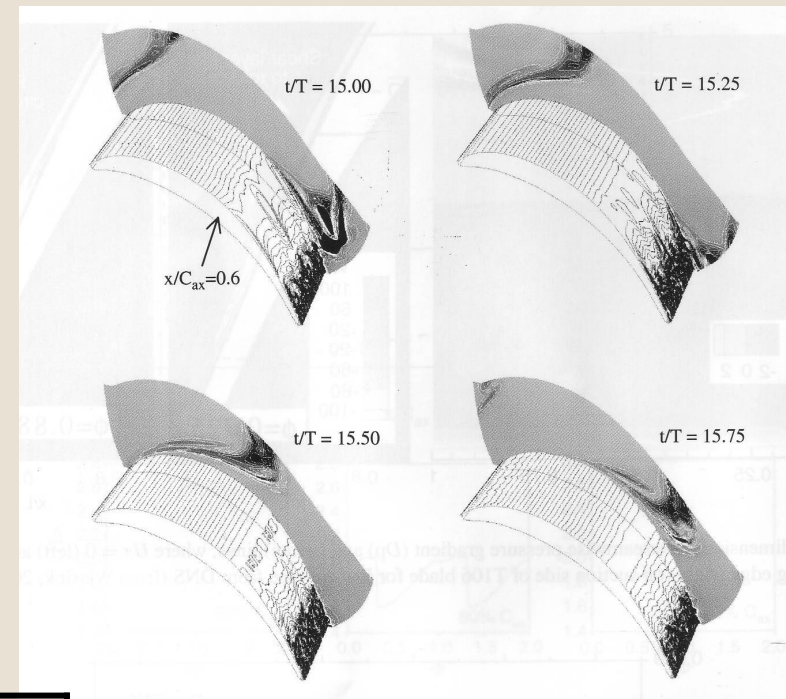


# Transition on T106 LPT blade ( $Re = 148000$ )



LES of  
Michelassi et al (2003)

- 10 mio grid points  
Dynamic SGS model
- Isolines of vertical velocity
- Span/Chord  $\sim 0.15$

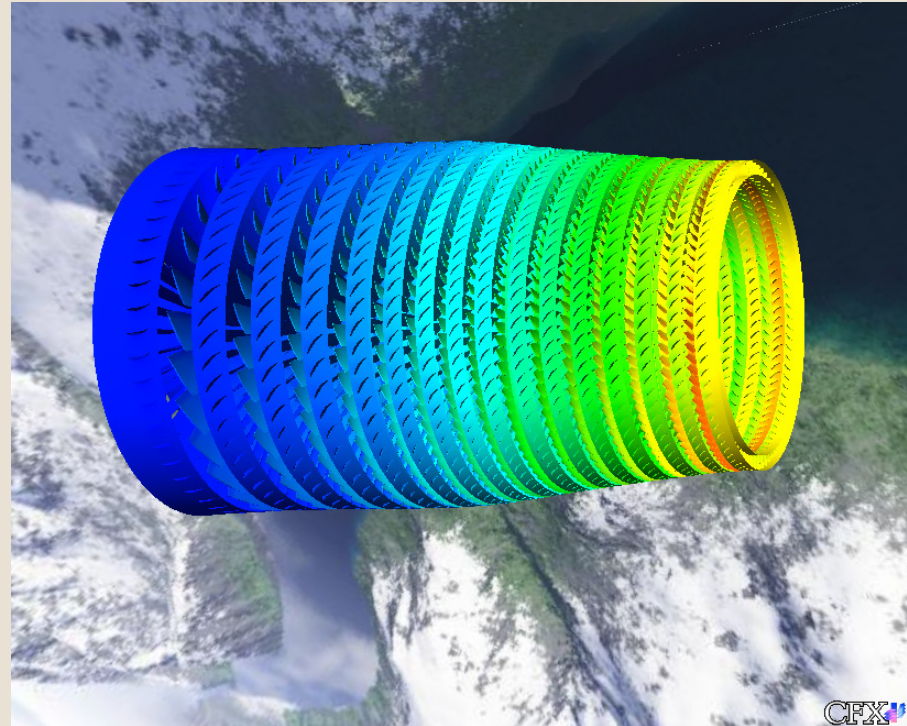
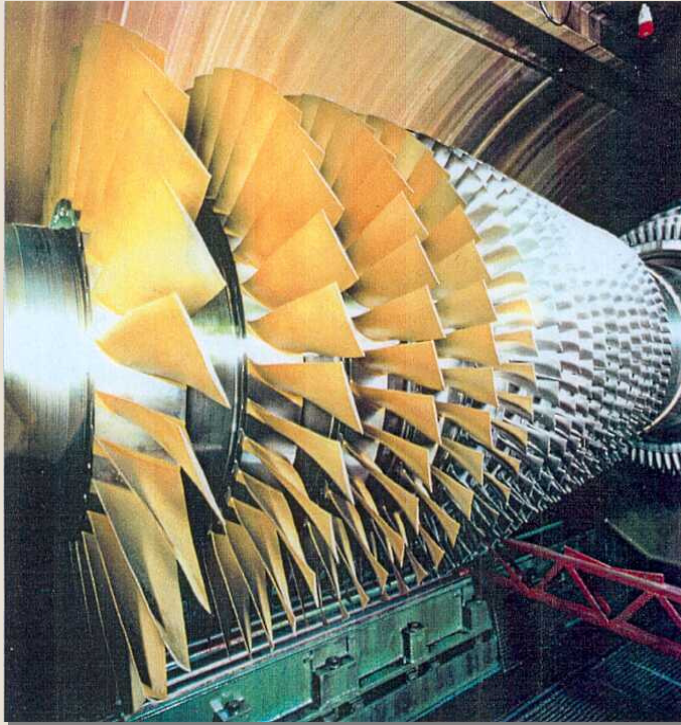


Full 3D	LES	Rans
Nodes	$\sim 150-300 \times 10^6$	$\sim 3-5 \times 10^6$
Dt- steady	$\sim 10^5$	$\sim 10^2$
Dt – unsteady	$\sim 2 \times 10^5$	$\sim 2 \times 10^3$

## LES/RANS Ratio

Ratio steady	$\sim 50,000$
Ratio unsteady	$\sim 5,000$

# Siemens 15 STAGE AXIAL COMPRESOR



- 15 rows x ~ 60 blades ~ 1000 blades
- Optimization (x 100 – 1000 configurations)

# Transition Modelling: Status Quo in Engineering



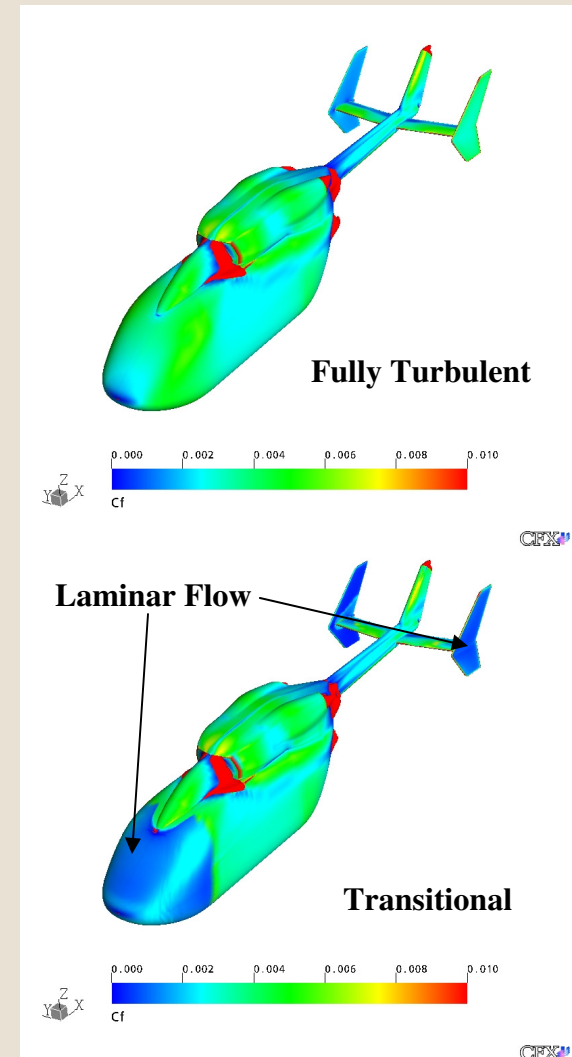
- **Low-Re models (only bypass transition)**
  - Based on transport equations for e.g.  $k$  and  $\varepsilon$  (compatible with modern CFD codes)
  - Cannot be calibrated independently of viscous sublayer model
  - Poor accuracy and robustness – not used in industry
- **$e^N$  method (only natural transition)**
  - Very accurate predictions for 2D airfoils (low FSTI)
  - N-S codes are not accurate enough to evaluate stability equations
  - Extension to generic 3D flows very difficult (impossible?)
  - Cannot account of non-linear effects (e.g. high FSTI, roughness)
- **Correlation based model**
  - Reasonably accurate
  - Correlations can be found for many different transition mechanisms (e.g. FSTI,  $dp/dx$ , Roughness)
  - Not compatible with 3D flows and unstructured/parallel CFD codes – non-local formulation

**Goal – correlation based model using transport equations**

# Transition Model Requirements



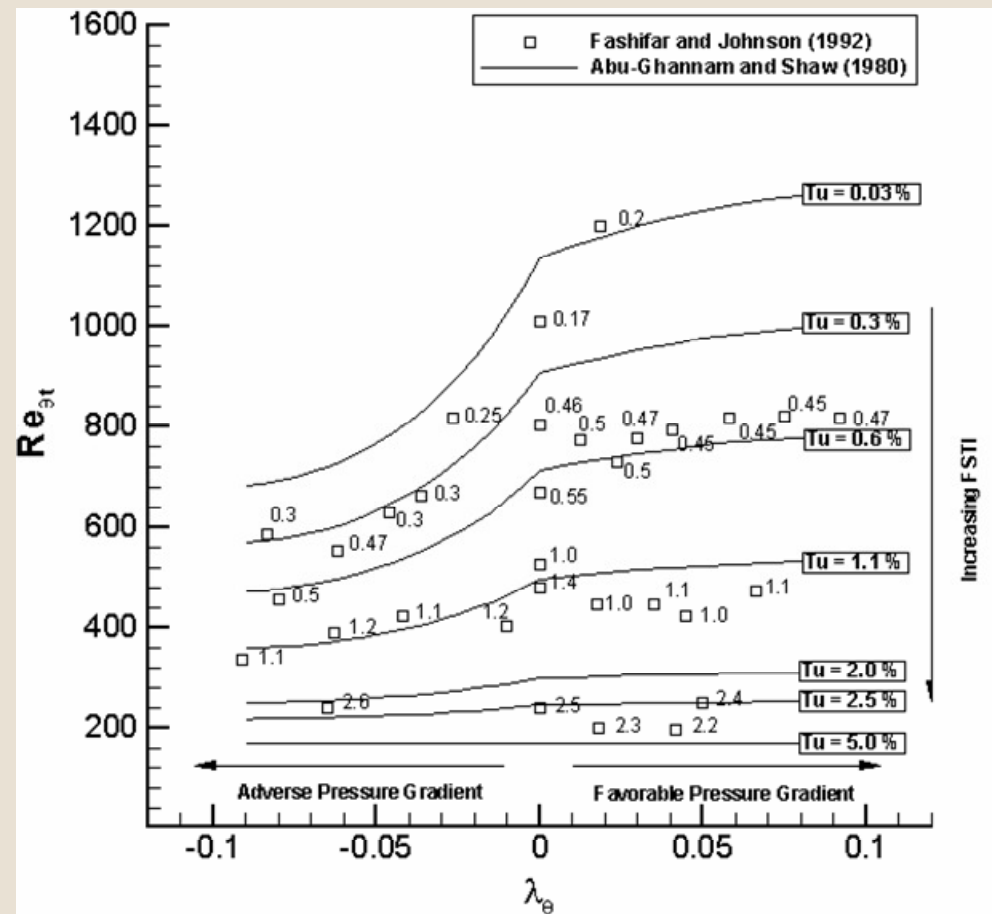
- **Compatible with modern CFD code:**
  - Unknown application
  - Complex geometries
  - Unknown grid topology
  - Unstructured meshes (no search directions)
  - Parallel codes – domain decomposition
- **Requirements:**
  - Absolutely no search algorithms
  - Absolutely no integration along lines
  - Local formulation
  - Different transition mechanisms
  - Robust
  - No excessive grid resolution



# Transition Onset Correlations



- Transition onset is affected by:
  - Free-stream turbulence turbulence intensity (FSTI)
  - Pressure gradients ( $\lambda_\theta$ )
  - Separation
  - Reynolds number ( $Re_\theta$ )
  - Mach number
  - Freestream length scale
  - Surface conditions:
    - Roughness
    - Temperature
    - Curvature
  - The history of the above parameters



$$Re_\theta = f(Tu, \lambda_\theta)$$

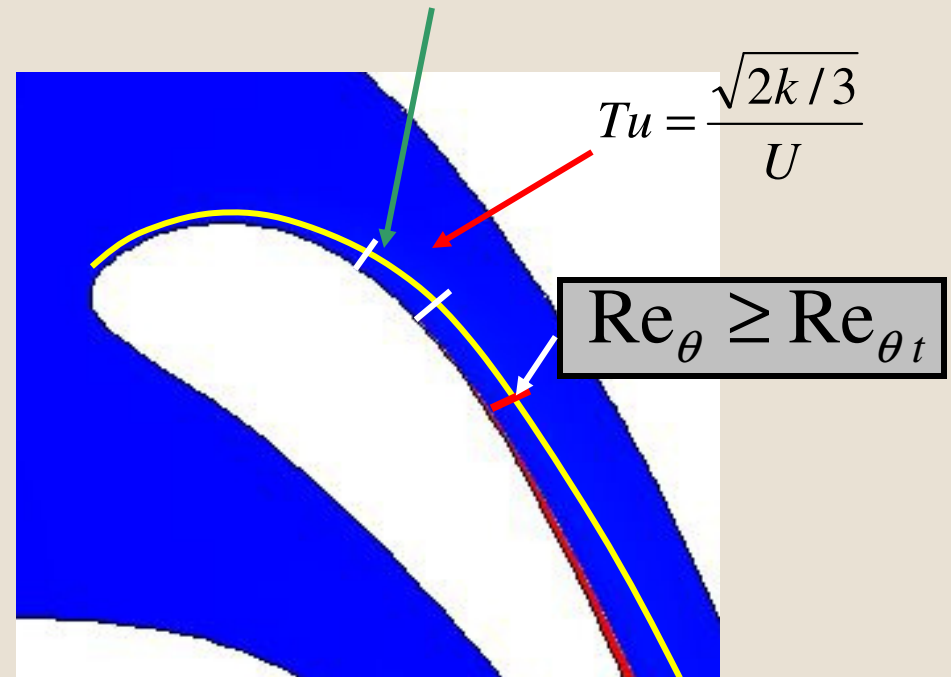


# Non-local formulations



- Transition onset:
  - Compute  $Re_{\theta}$  for all laminar bl-profiles and compare with  $Re_{\theta t}$
- Length of transition
  - Trigger turbulence model with ramp-function
- Correlation
  - Evaluated at edge of the boundary layer

$$\theta = \int_0^{\delta} \frac{u}{U} \left( 1 - \frac{u}{U} \right) dy \rightarrow Re_{\theta} = \frac{\rho U \theta}{\mu}$$



$$Re_{\theta t} = f(Tu, \lambda_{\theta})$$

# Transport Equation for Intermittency $\gamma$

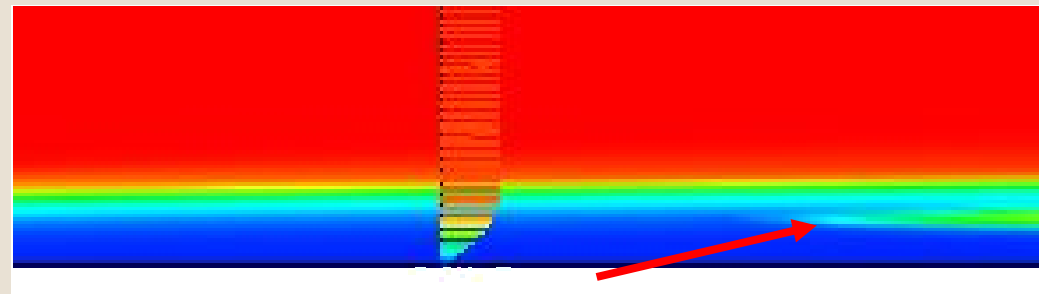


$$\frac{\partial(\rho\gamma)}{\partial t} + \frac{\partial(\rho U_j \gamma)}{\partial x_j} = P_\gamma - E_\gamma + \frac{\partial}{\partial x_j} \left[ \left( \mu + \frac{\mu_t}{\sigma_f} \right) \frac{\partial \gamma}{\partial x_j} \right]$$

## Transition Sources

$$P_\gamma = F_{length} c_{a1} \rho S [\gamma F_{onset}]^{0.5} (1 - c_{e1} \gamma) \quad \mathbf{F_{onset}} \text{ transition onset when } \text{Re}_\theta \geq \text{Re}_{\theta t}$$

$$E_\gamma = c_{a2} \rho \Omega \gamma F_{turb} (c_{e2} \gamma - 1) \quad \mathbf{F_{length}} \text{ length of transition}$$



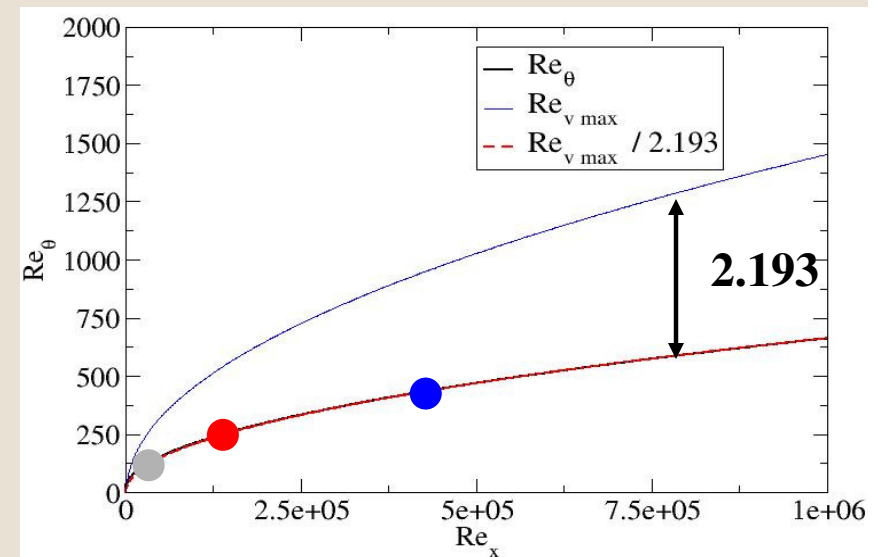
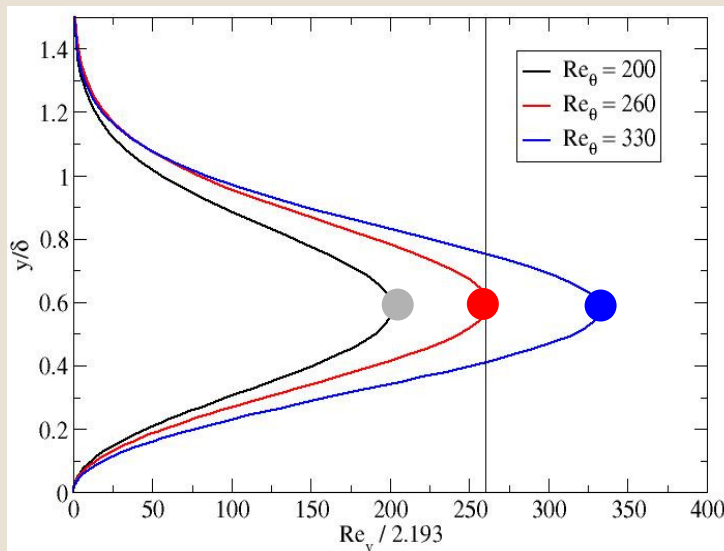
**Onset**

# New Idea: Vorticity Reynolds Number



$$Re_v = \frac{\rho y^2}{\mu} \left| \frac{\partial u}{\partial y} \right|$$

Blasius Boundary Layer



$$(Re_v)_{\max} = (Re_\theta) \cdot 2.193$$

- Maximum value of  $Re_v$  in the B.L. is proportional to  $Re_\theta$
- Can relate  $Re_{\theta t}$  from correlation to  $Re_v$
- Allows empirical correlations to be used with 3d, unstructured parallel solvers

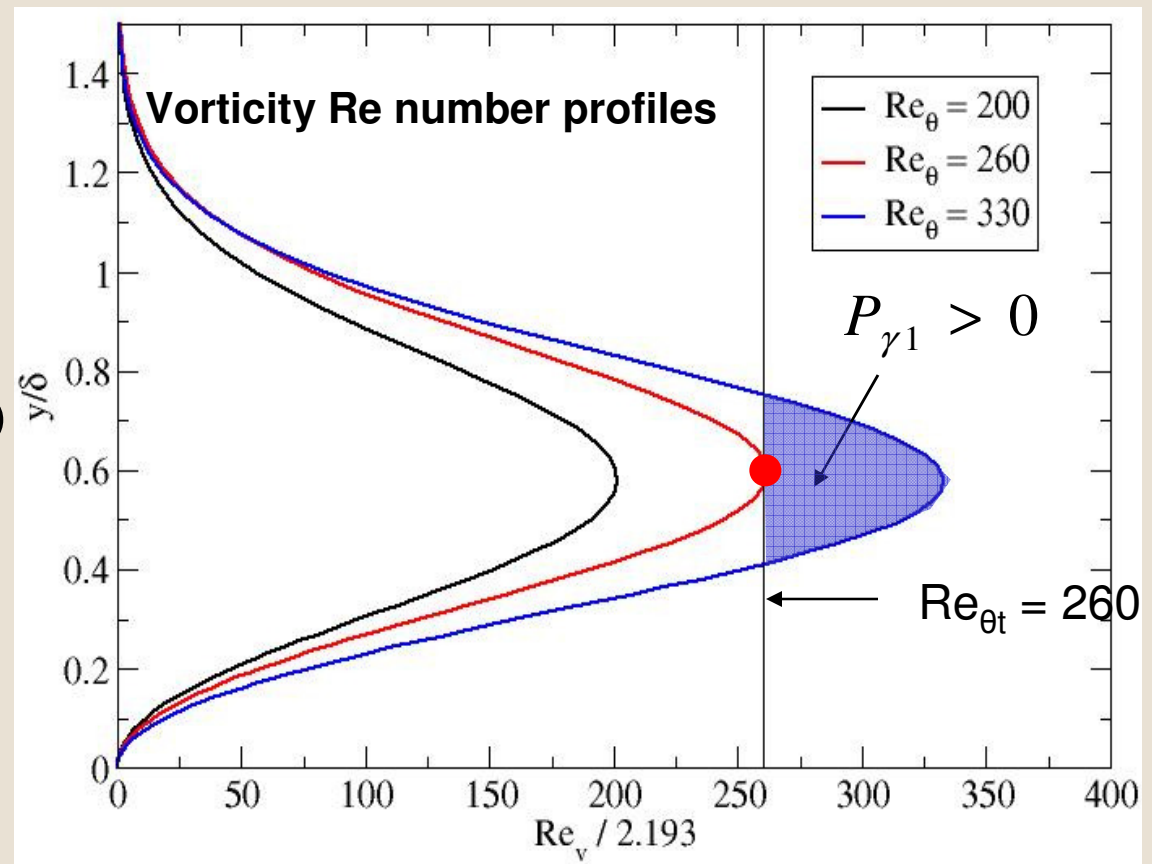
# Production of Intermittency



$$Re_v = \frac{\rho y^2}{\mu} \left| \frac{\partial u}{\partial y} \right|$$

$$F_{onset} \sim \max\left(\frac{Re_v}{2.193 Re_{\theta t}} - 1, 0\right)$$

$$Re_{\theta t} = f(Tu, \lambda_{\theta})$$





# Modification to SST Turbulence Model



$$\frac{\partial}{\partial t}(\rho k) + \frac{\partial}{\partial x_j}(\rho U_j k) = \tilde{P}_k - \tilde{D}_k + \frac{\partial}{\partial x_j} \left( (\mu + \sigma_k \mu_t) \frac{\partial k}{\partial x_j} \right)$$

$$P_k = \mu_t S^2$$

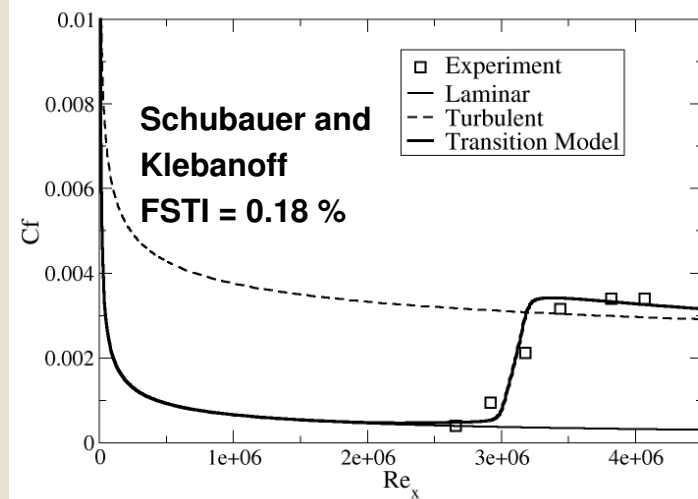
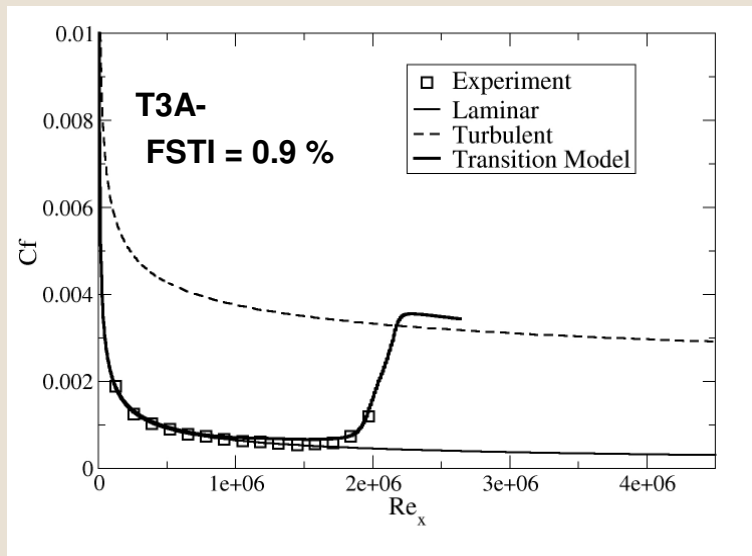
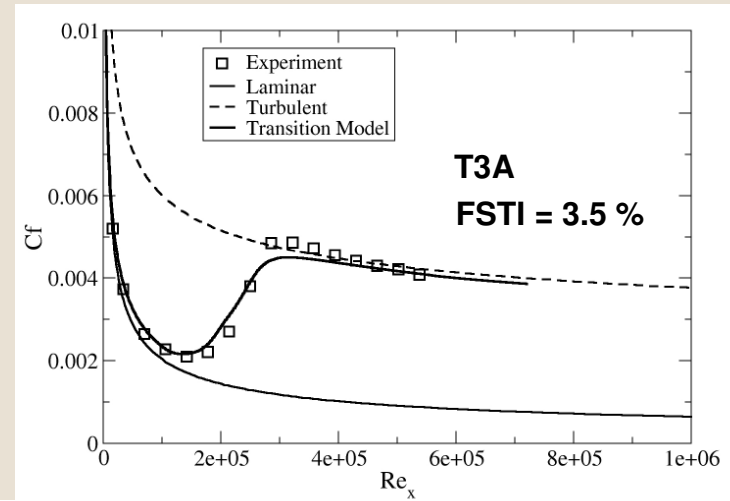
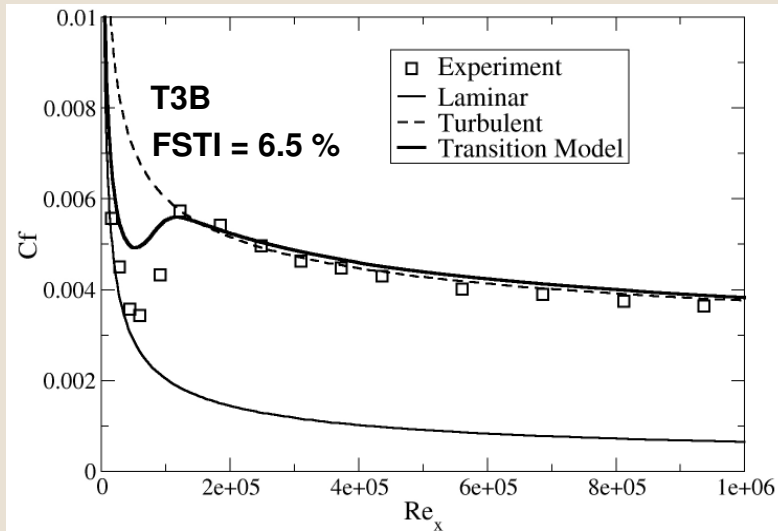
$$D_k = \beta^* \rho k \omega$$

$$\tilde{P}_k = \gamma_{eff} P_k$$

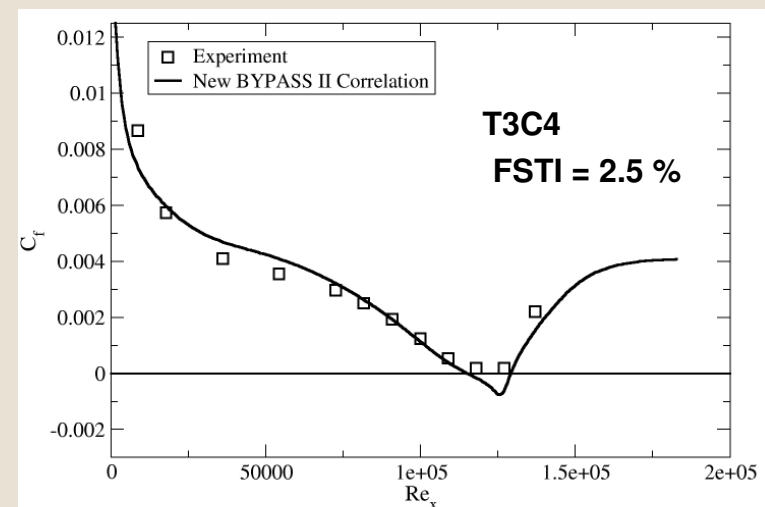
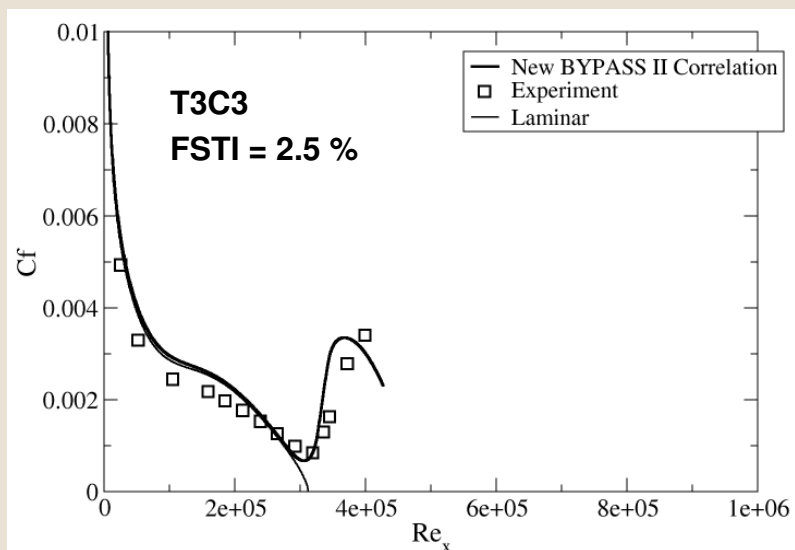
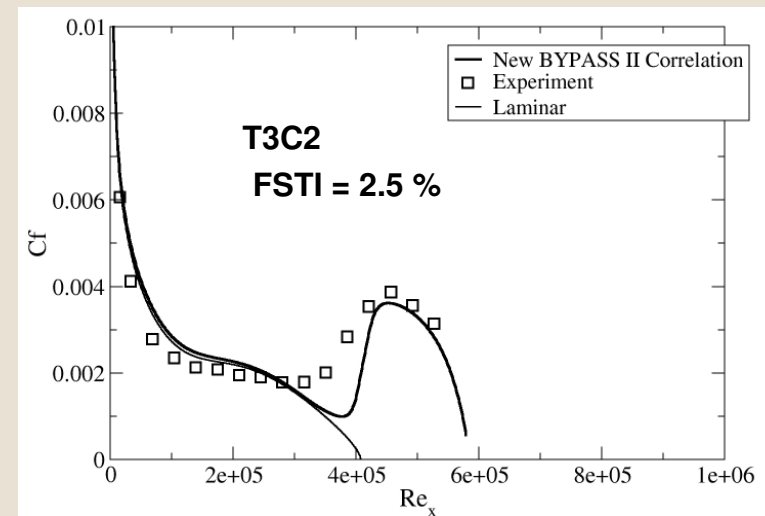
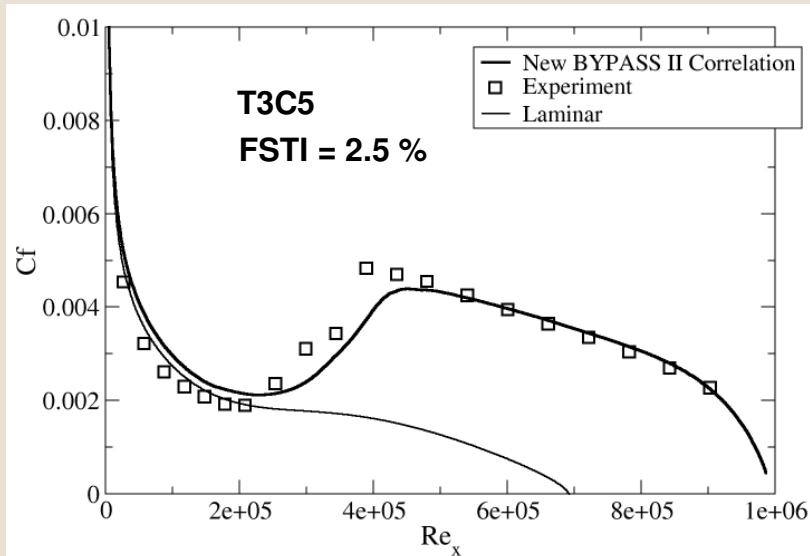
$$\tilde{D}_k = \min(\max(\gamma_{eff}, 0.1), 1.0) D_k$$

***S – invariant form of strain-rate***

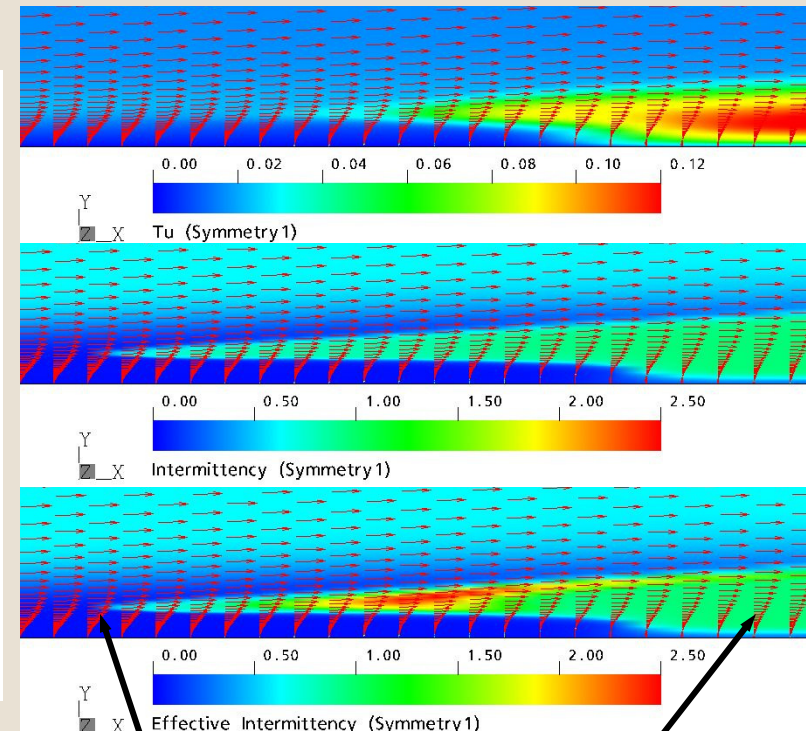
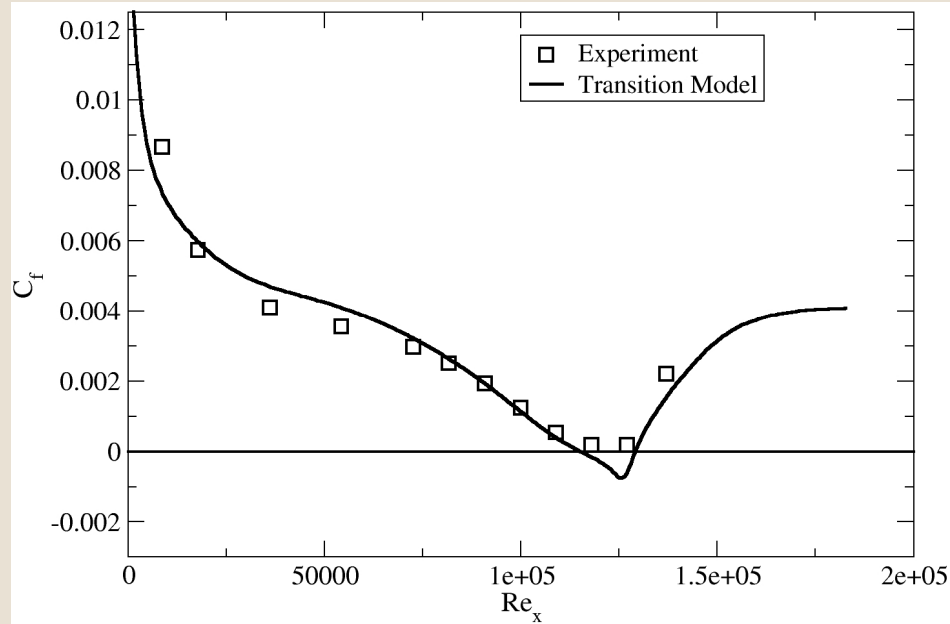
# Flat Plate Results: $dp/dx=0$



# Flat Plate Results: $dp/dx$ (change in Re number)



# Separation Induced Transition



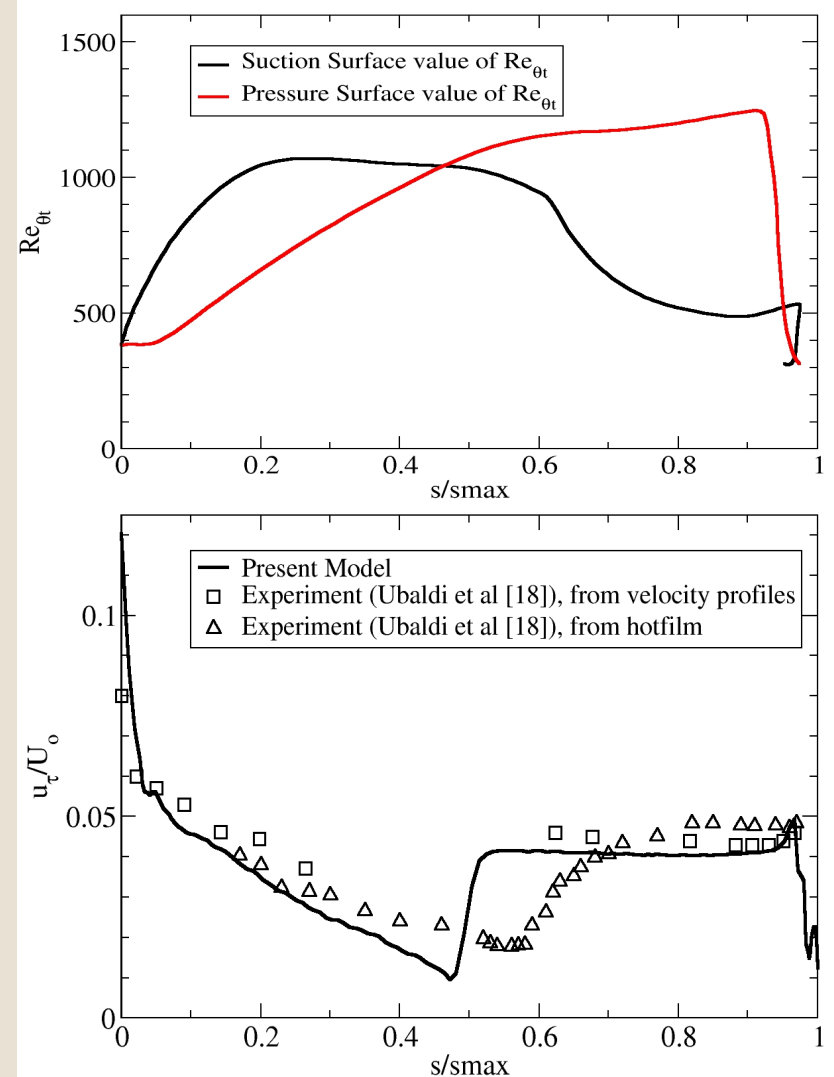
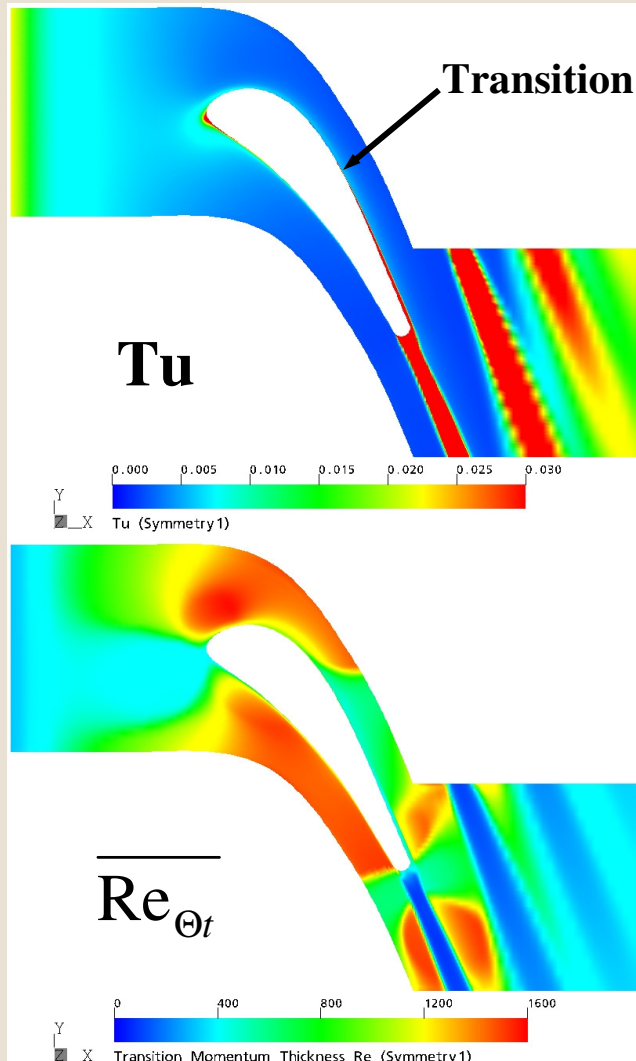
Laminar  
Separation

Turbulent  
Reattachment

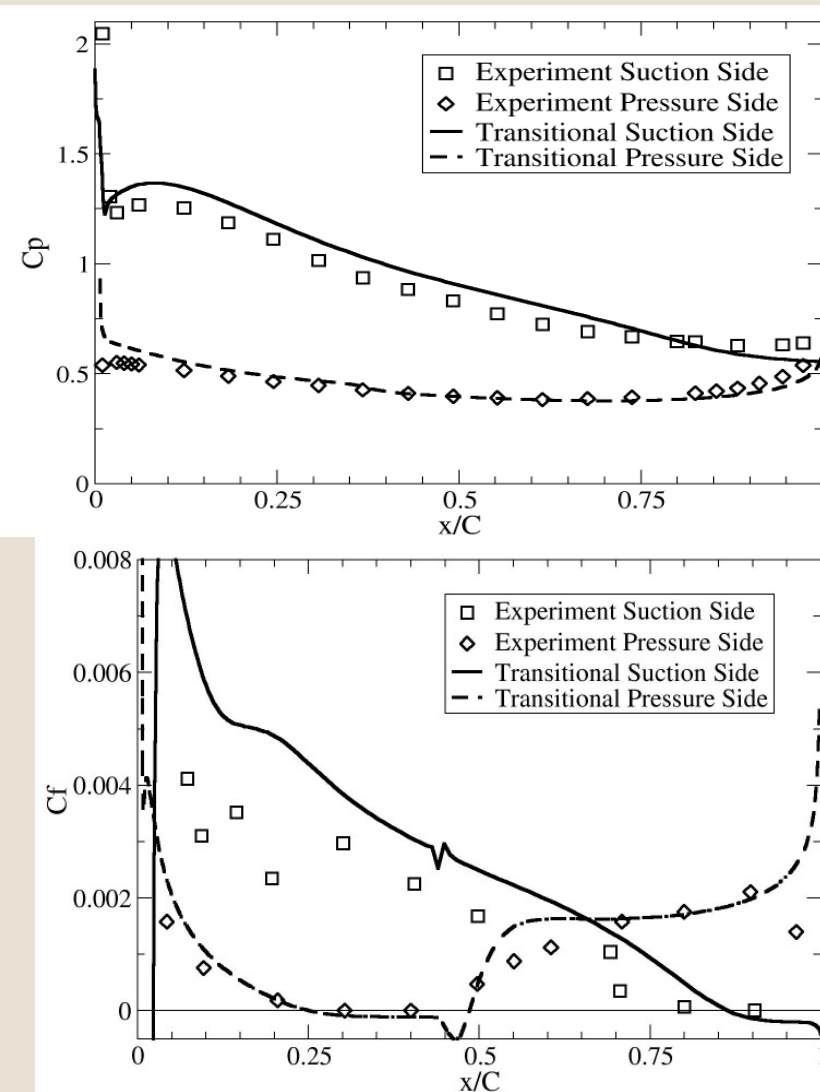
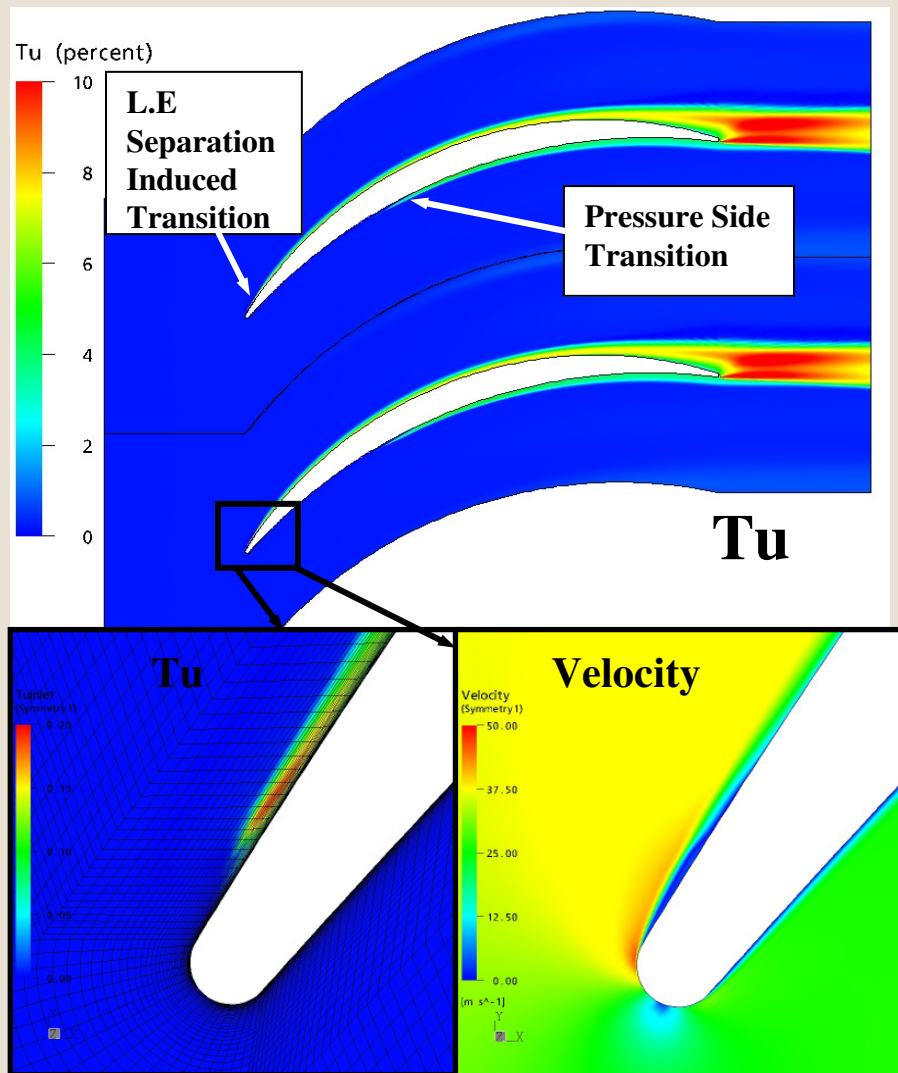
- Previous versions predicted separation induced transition too late
- Solution: Allow intermittency to increase above 1.0 in laminar separation bubbles



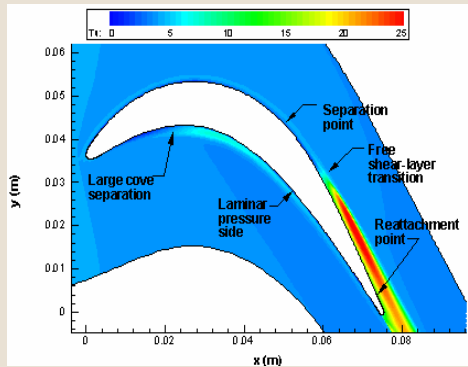
# Test Cases: Genoa Turbine Blade



# Test Cases: Zierke (PSU) Compressor Blade

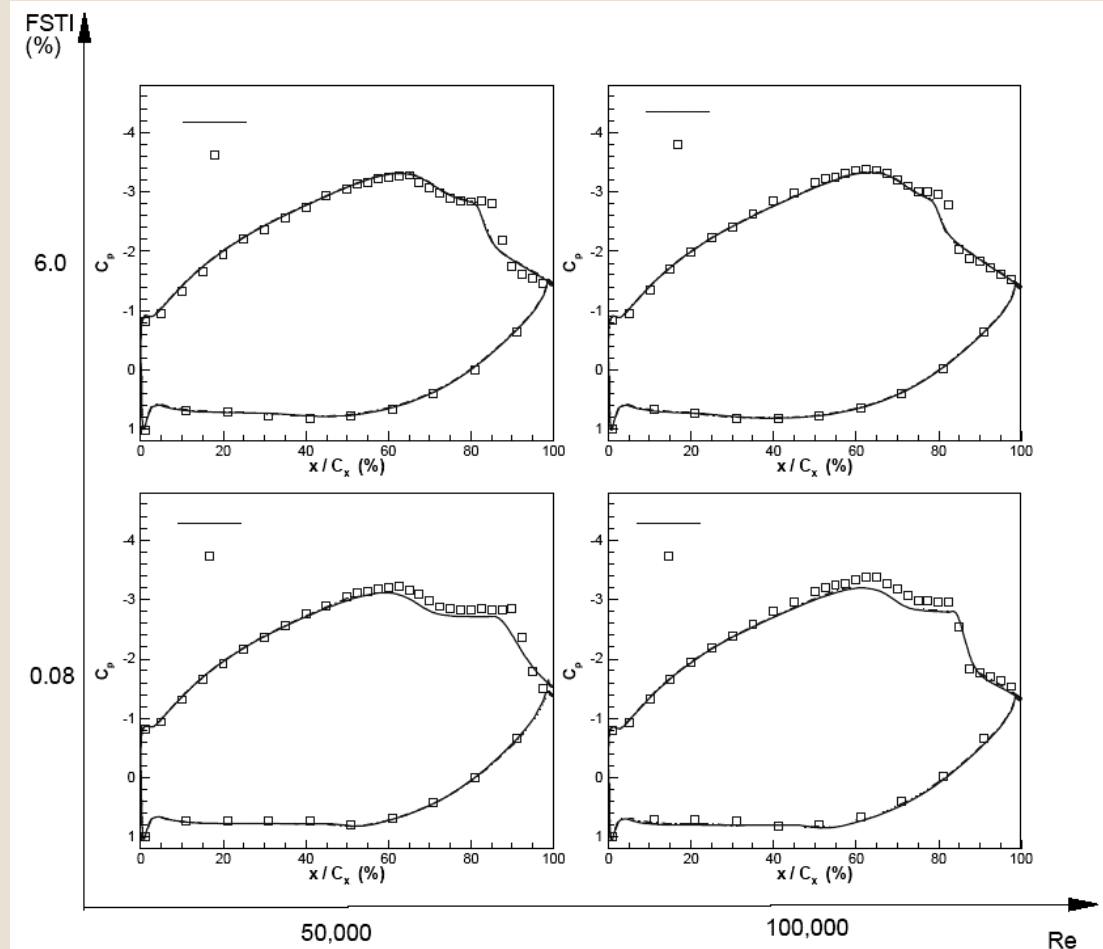


# Separation Induced Transition On an LP-Turbine



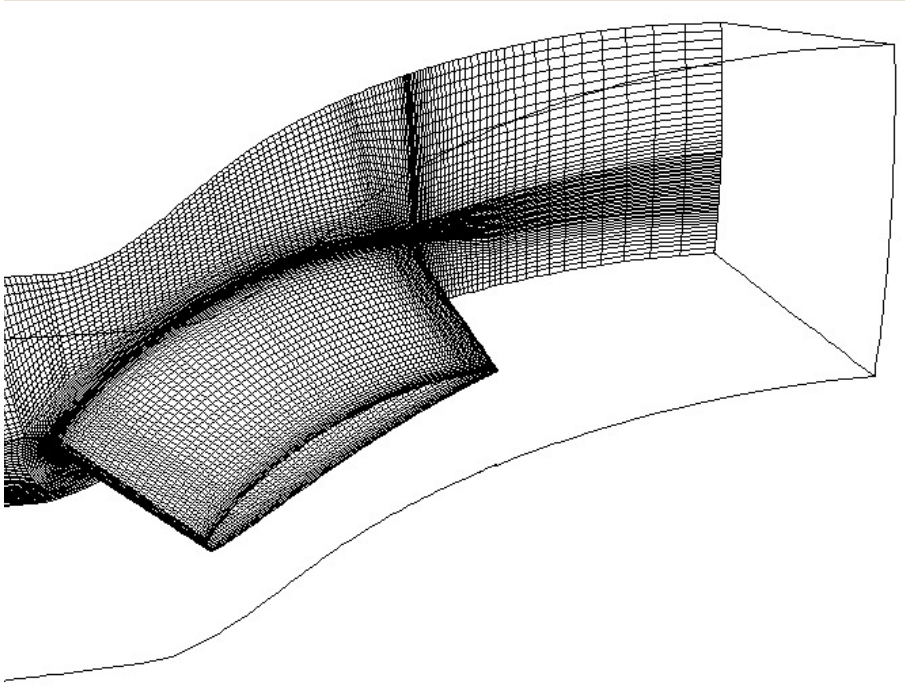
Increasing FSTI

- Pratt and Whitney Pak-B LP turbine blade
- $Re_x = 50\,000, 75\,000$  and  $100\,000$
- $FSTI = 0.08, 2.25, 6.0$  percent
- Computations performed by Suzen and Huang, Univ. of Kentucky



Increasing  $Re_x$

# Test Cases: 3D RGW Compressor Cascade



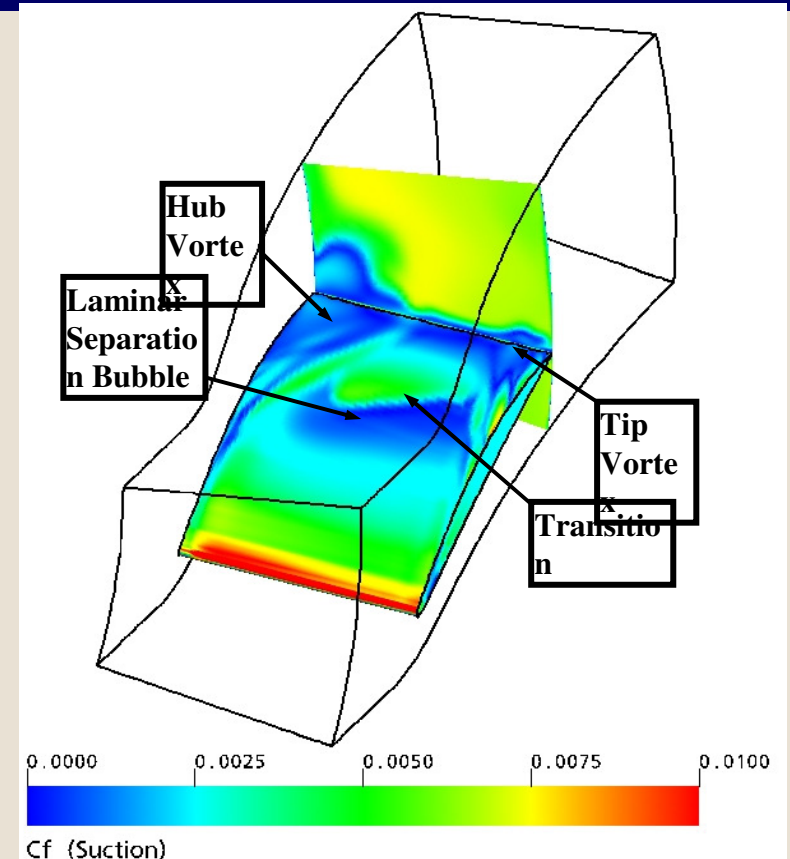
**RGW Compressor (RWTH Aachen)**

**FSTI = 1.25 %**

**$Re_x = 430\,000$**

**Loss coefficient,  $(Y_p) = 0.097$**

**$Y_p = (p_{inlet} - p_{outlet})/p_{dynoutlet}$**

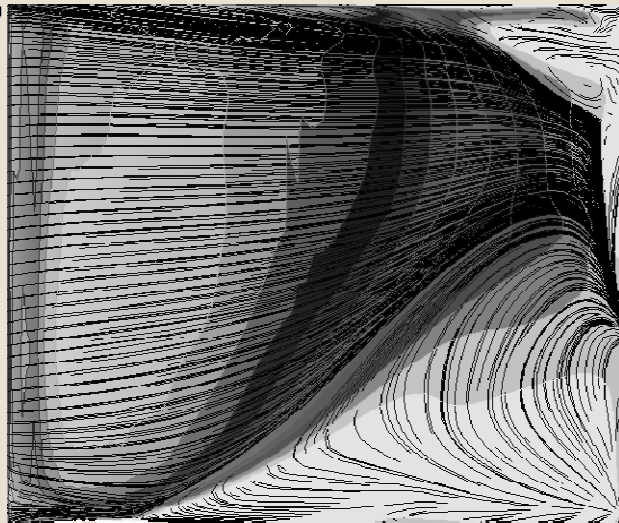


Schulz, H.D., Gallus, H.D., 1988, "Experimental Investigation of the Three-Dimensional Flow in an Annular Compressor Cascade", *ASME Journal of Turbomachinery*,

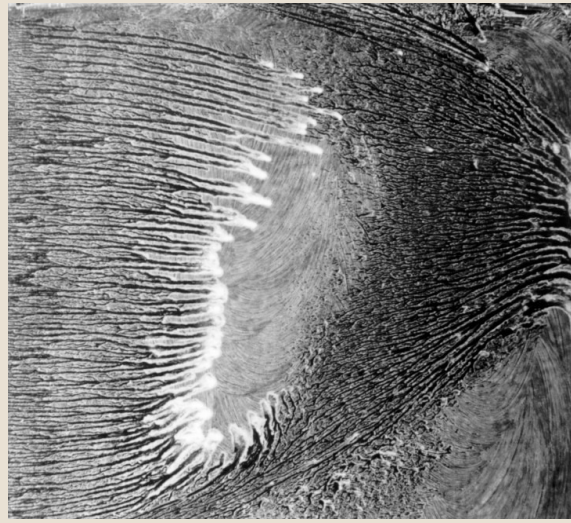




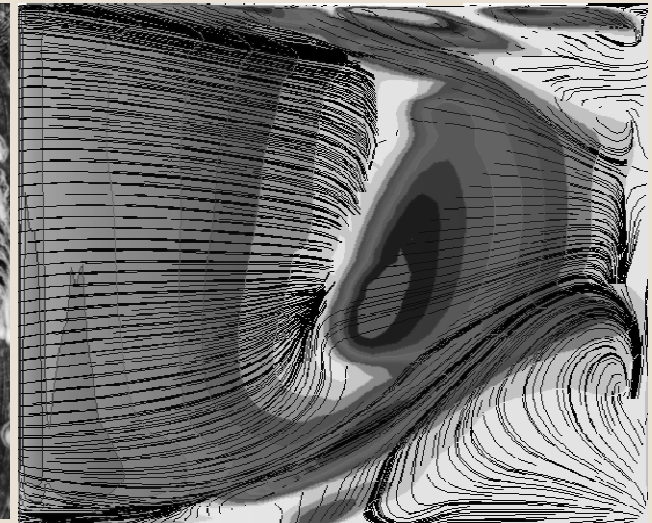
# Test Cases: 3D RGW Compressor Cascade



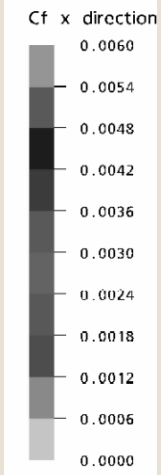
**Fully Turbulent**  
 $Y_p = 0.19$



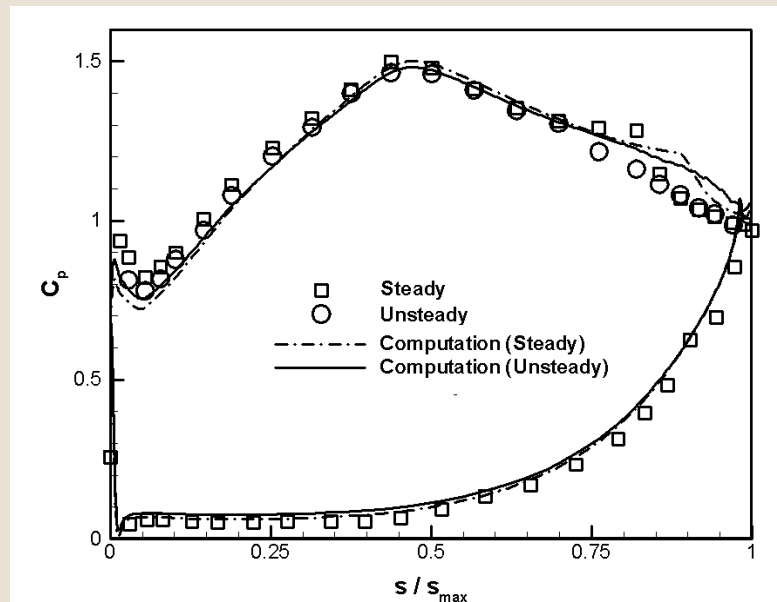
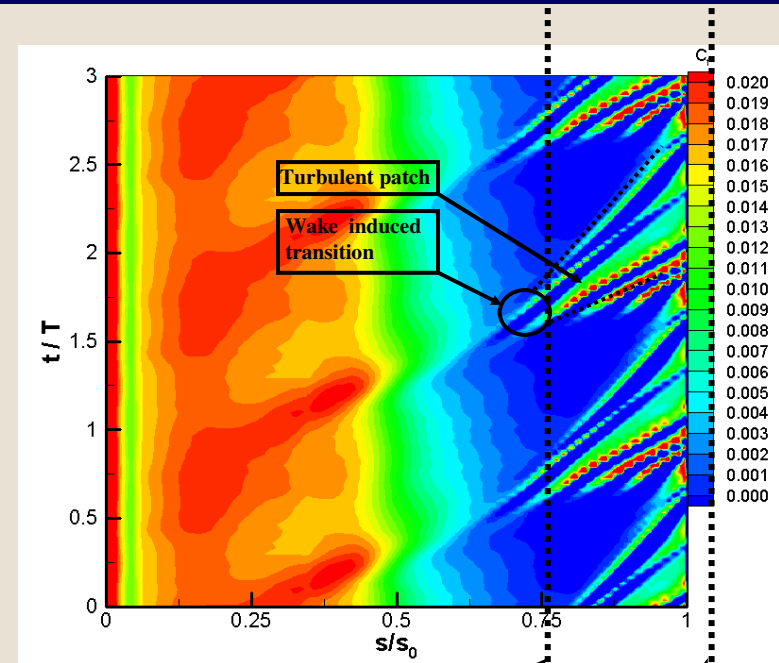
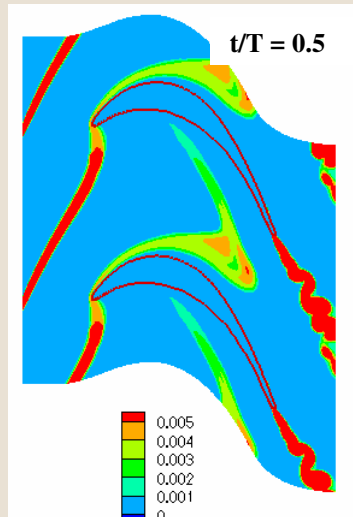
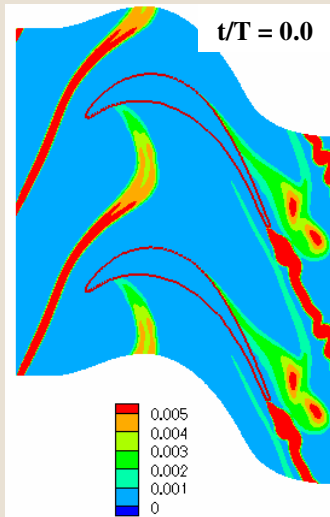
**Experimental Oil Flow**  
 $Y_p = 0.097$



**Transition Model**  
 $Y_p = 0.11$



# Unsteady Wake Induced Transition (George Huang)

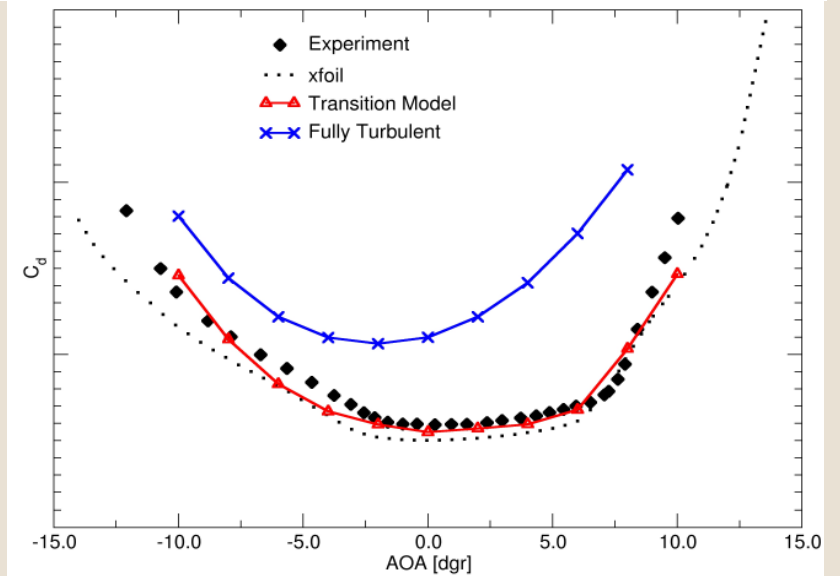
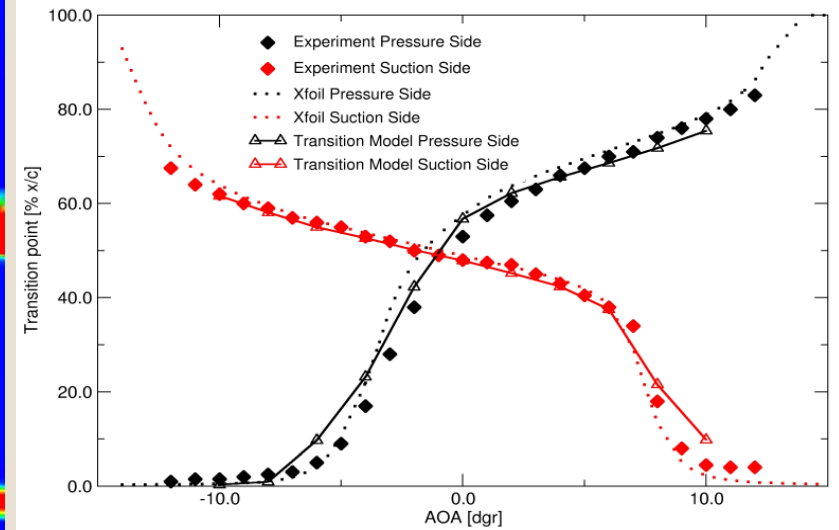
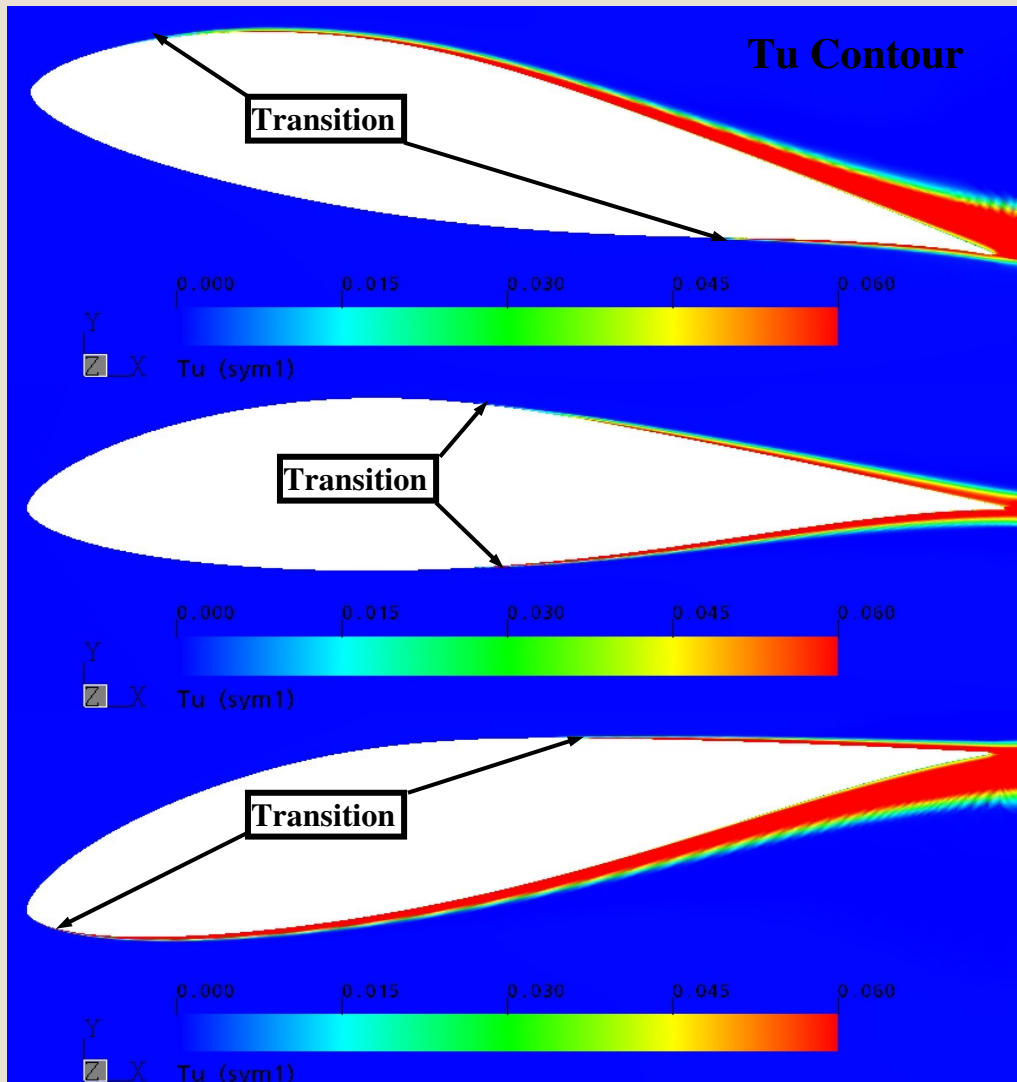


Steady Laminar  
Separation

Steady Turbulent  
Reattachment

Stieger et al. (2003)

# Wind Turbine Airfoil

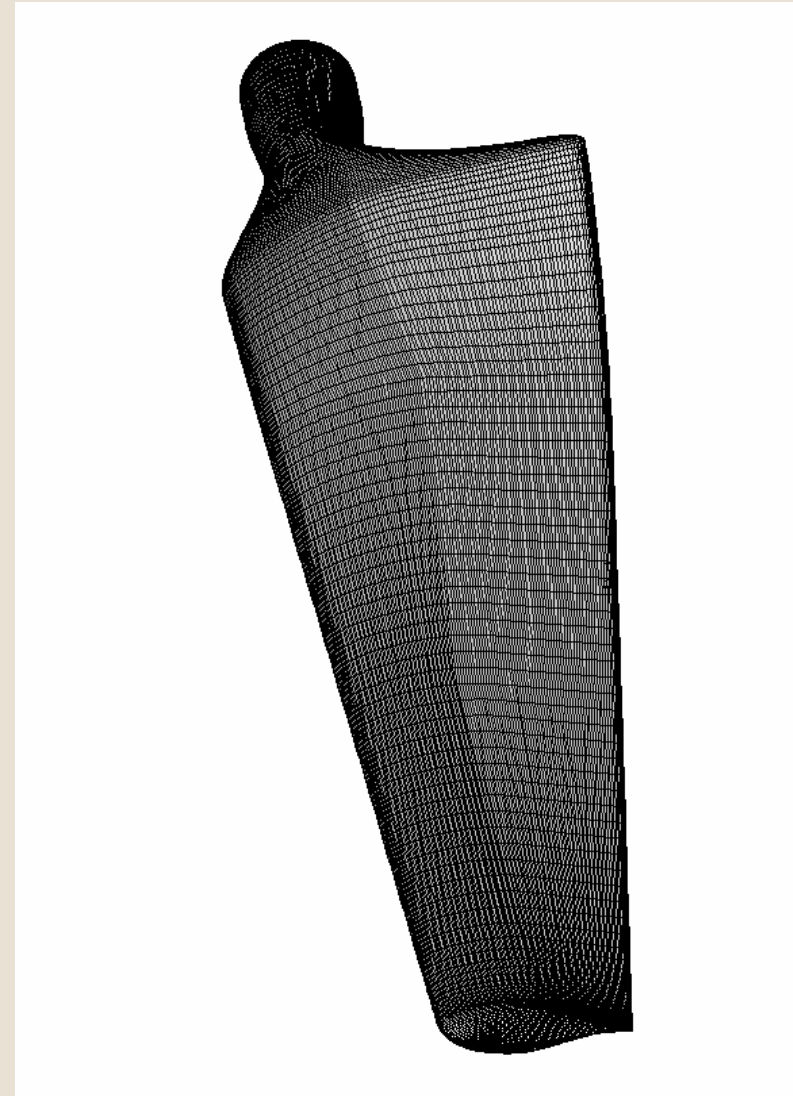


## 3D NREL Wind Turbine (S809 Airfoil)



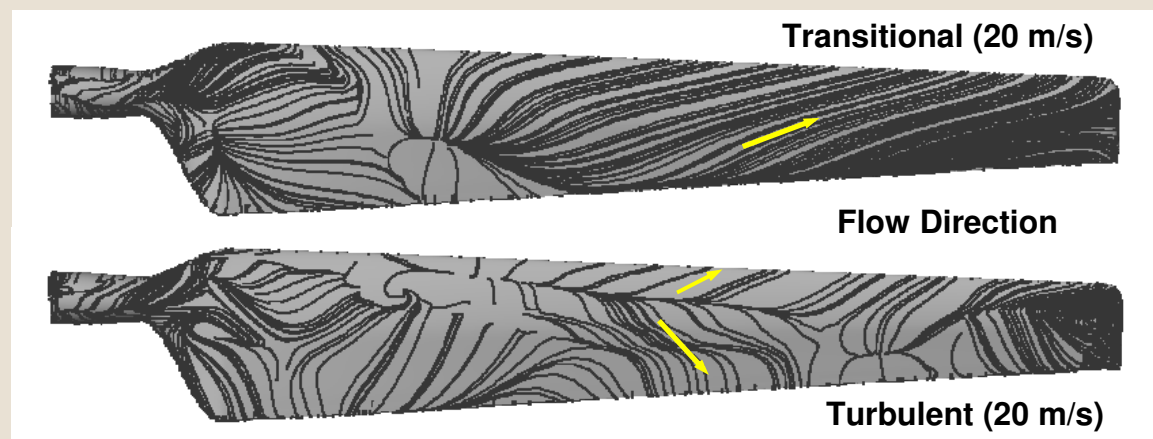
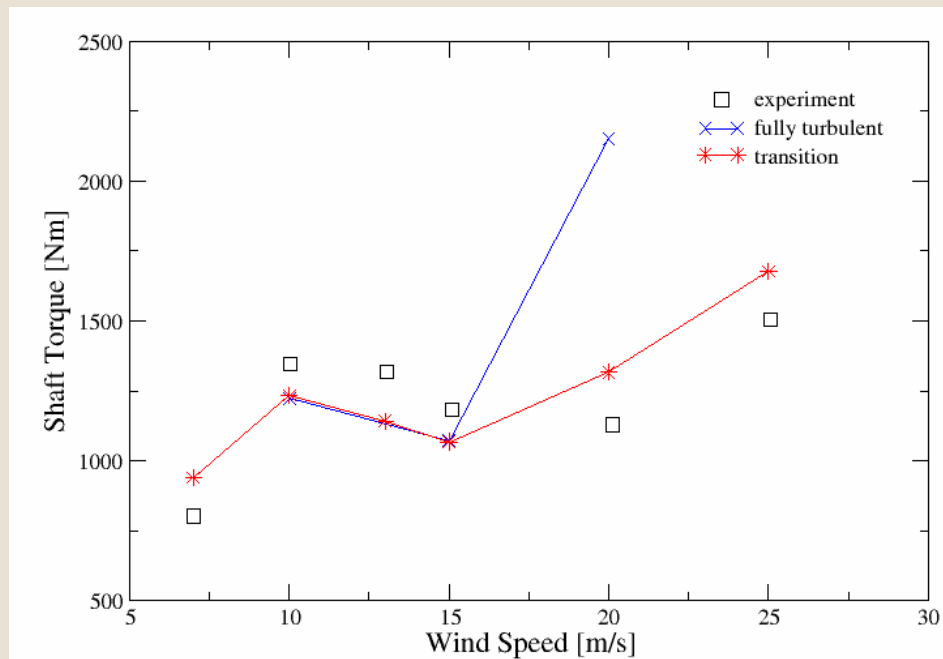
- Full 3D Wind Turbine
- Wind Speeds = 7 to 25 m/s
- S809 airfoil profile for the NREL Phase IV full wind turbine experiment, (Simms, 2001)
- All CFD computations performed with ANSYS CFX 10
  - Transitional and Fully Turbulent
  - Grid = 10 million Nodes
  - Each run made overnight on a 16 CPU Linux cluster
  - Max  $y^+ = 1$

Simms, D., Schreck, S., Hand, M., and Fingersh, L.J. (2001). "NREL Unsteady Aerodynamics Experiment in the NASA-Ames Wind Tunnel: A Comparison of Predictions to Measurements", *NREL Technical report, NREL/TP-500-29494*.





# NREL Wind Turbine: Shaft Torque

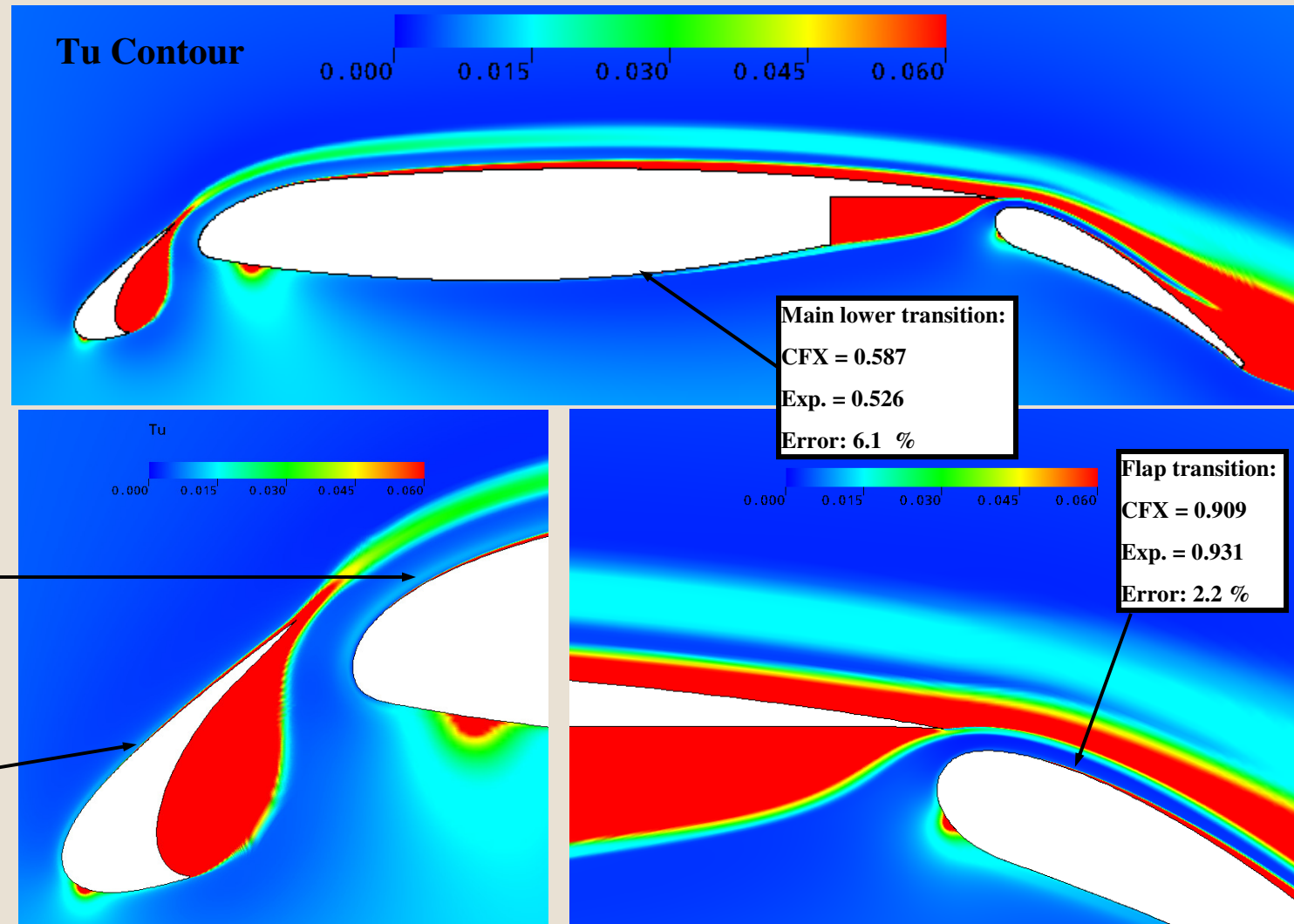


# McDonnell Douglas 30P-30N 3-Element Flap



Re = 9 million  
Mach = 0.2  
C = 0.5588 m  
AoA = 8°

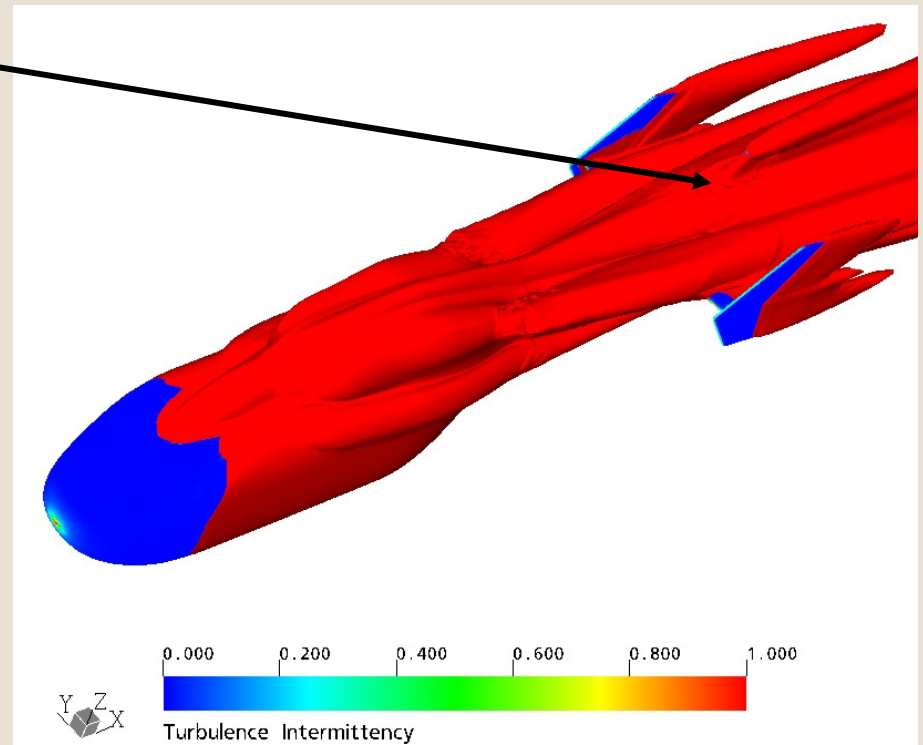
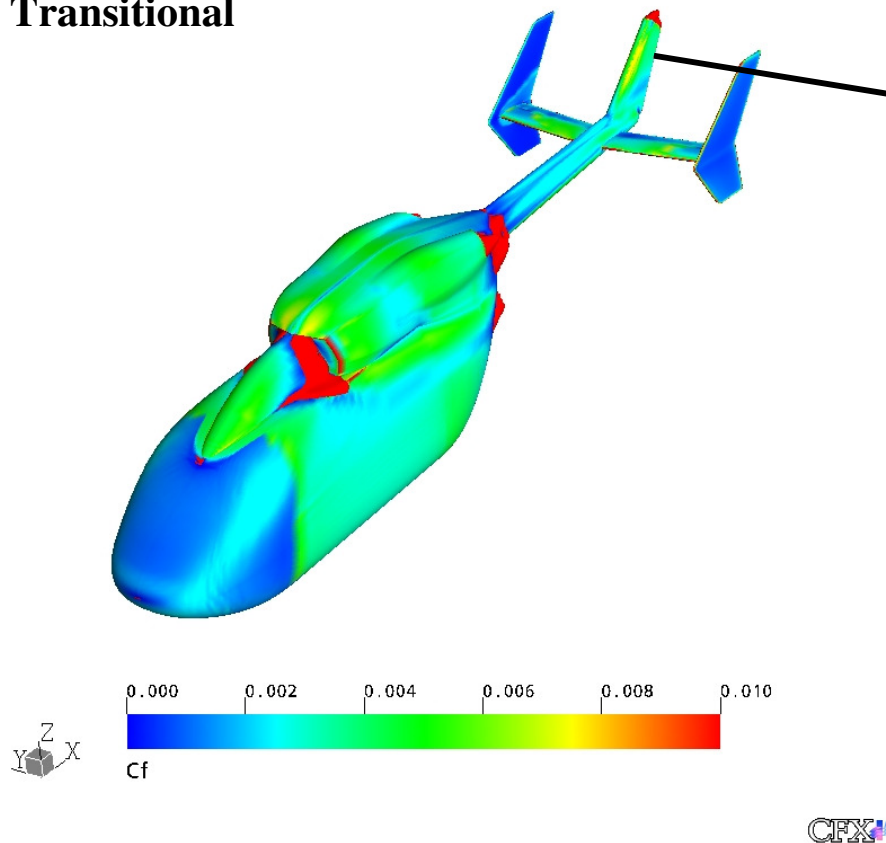
Exp. hot film  
transition  
location  
measured  
as  $f(x/c)$



# Eurocopter – Wake induced transition



Transitional



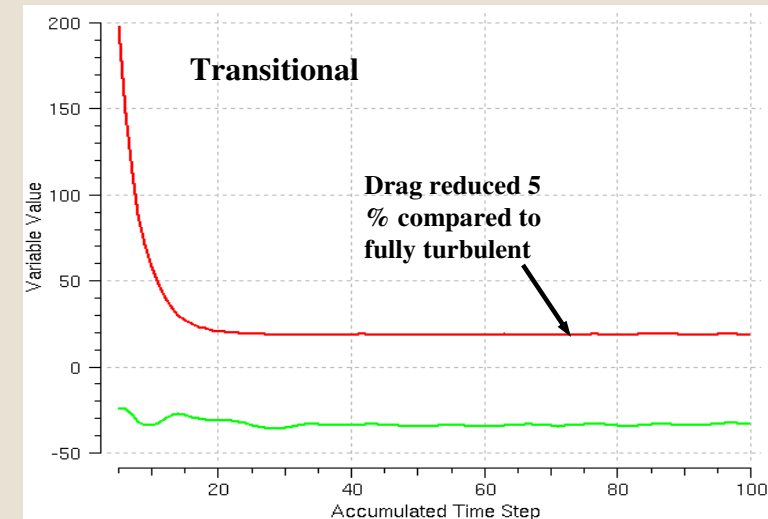
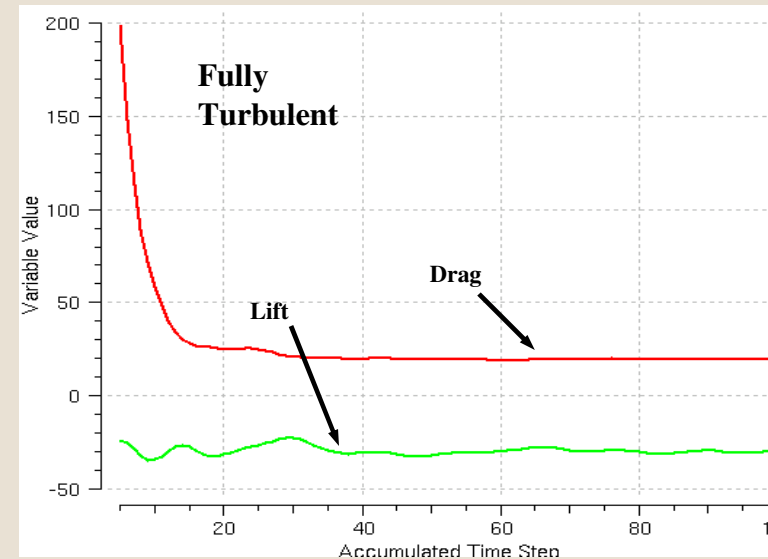
$Re_x = 30$  million



# Convergence and Cost of using the transition model



- Eurocopter configuration
- 6 million nodes
- Max  $y^+ = 1$
- 16 CPU's
- Total Additional CPU cost 17%
  - Discretization (High Res) 12%
  - Linear Solution 5%



# Team Alinghi in Action





# Wind Tunnel Facility



Wind Tunnel testing carried out at NRC's 9m x 9m wind tunnel in Ottawa (Canada)

- 1.5:1 model scale (to match Re)
- measure of force on the whole appendage and on each appendage component (keel, bulb, winglets)
- transition location study (with thermography)
- Study on the dependency on inflow turbulence levels



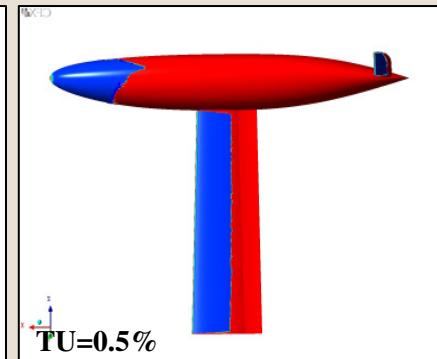
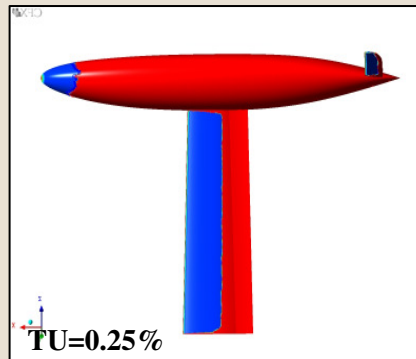
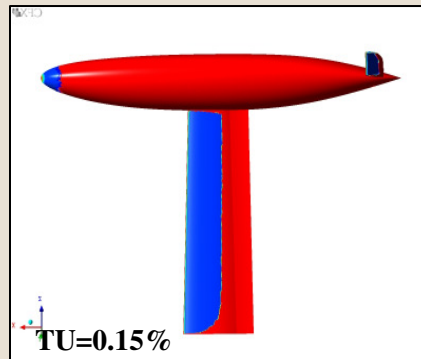
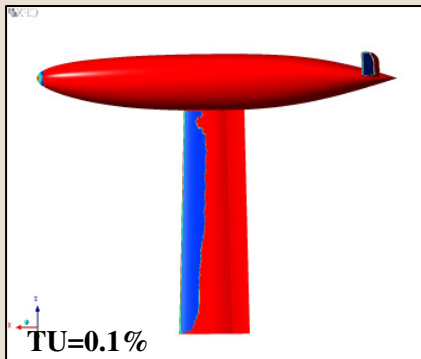
# Transition Modelling Calibration



Inflow turbulence intensity calibration:

- Comparison with high turbulence wind tunnel run
- Good force and transition location matching for TU=0.5%

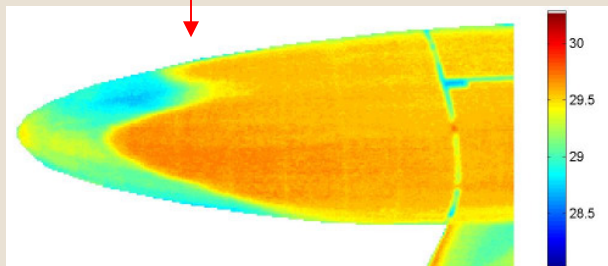
TU	Global Drag	Error	Bulb Lam %	Keel Lam %
0.10%	-0.0460	16%	18-26	57
0.15%	-0.0483	12%	8-12	57
0.25%	-0.0495	10%	5-6	54
0.50%	-0.0558	1%	2-2.5	29-36
Wind Tunnel	-0.0550		2	27-30



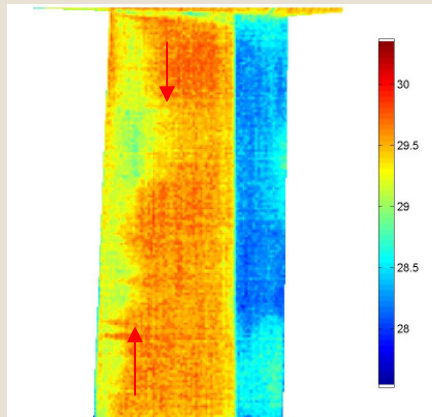
# Transition Modelling Testing



## Wind Tunnel Thermography

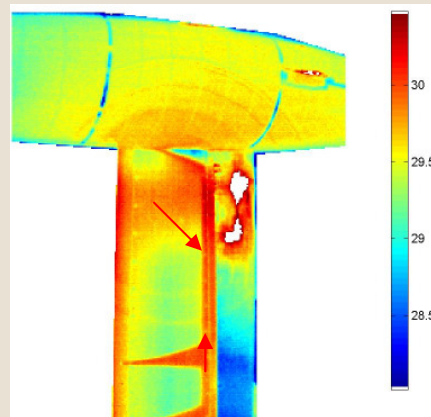


Bulb: 7%-15%

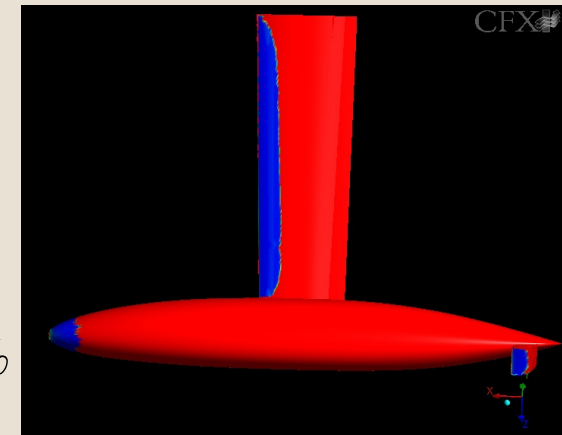


Keel (suction side): 12.5% - 23.5%

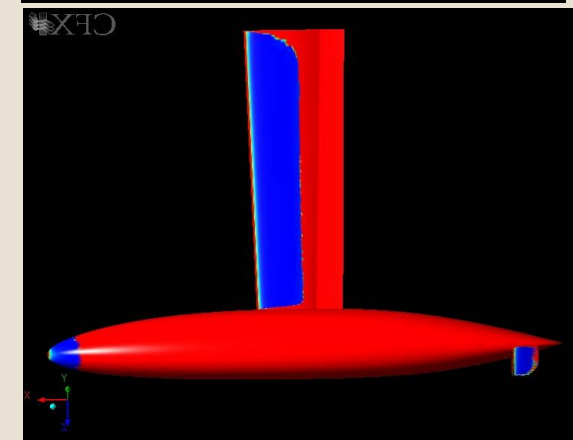
Keel (pressure side): 62%



## CFD Analysis



Bulb: 8%



Keel (suction side): 24%

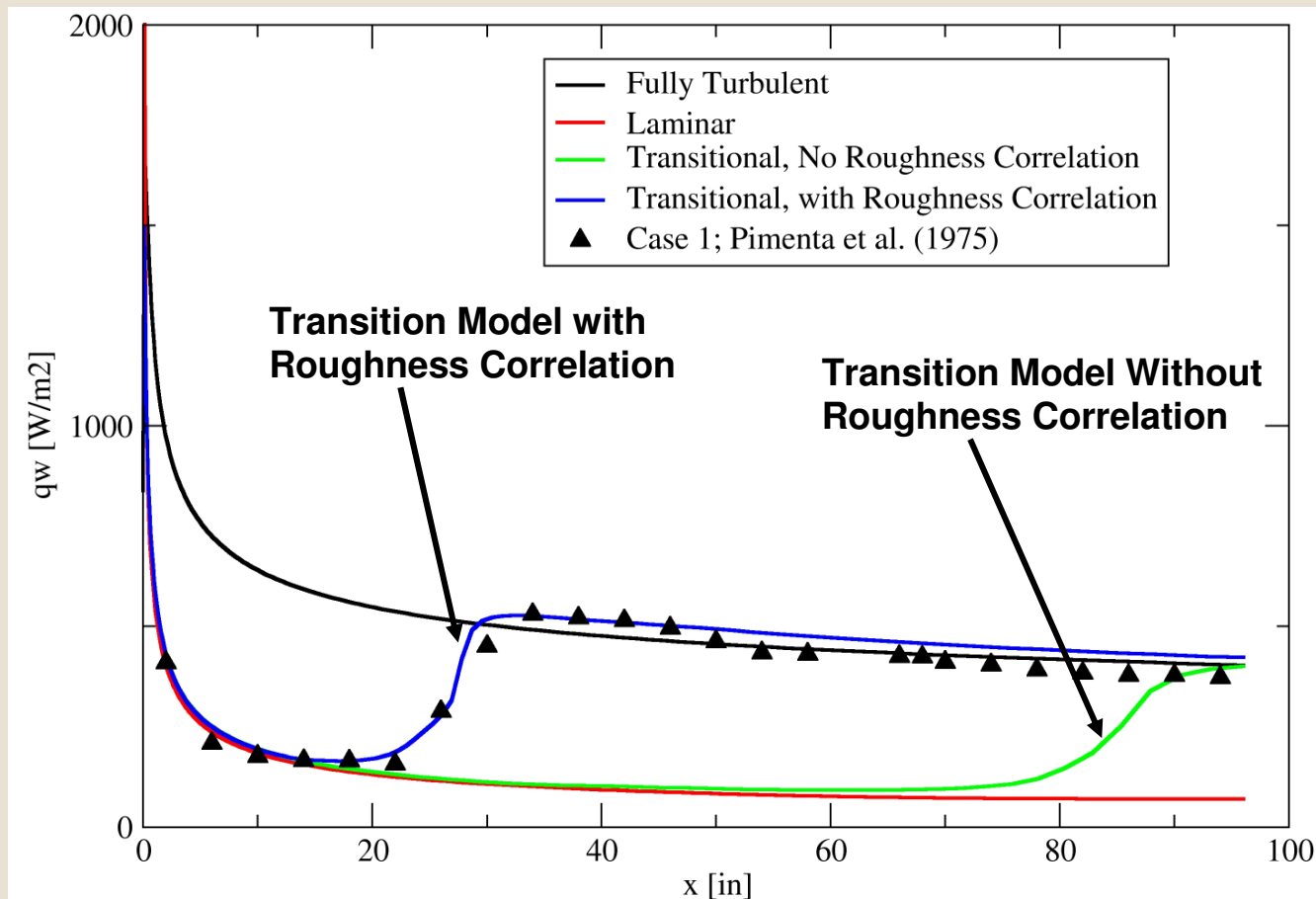
Keel (pressure side): 57%

- New correlation based transition model has been developed
  - Based strictly on local variables
  - Applicable to unstructured massively parallelized codes
- Onset prediction is completely automatic
  - User must specify correct values of inlet  $Tu$  and  $R_T$
- Validated for a wide range of 2-D and 3-D turbomachinery and aeronautical test cases – **one set of correlations for all flows**
- Strong potential that 1<sup>st</sup> Order effects of transition can be captured in everyday industrial CFD simulations
- Opens many new opportunities in industrial CFD –  
Implementation into Fluent 6.4 – Formula One teams ...

# Flat Plate Transition with Roughness

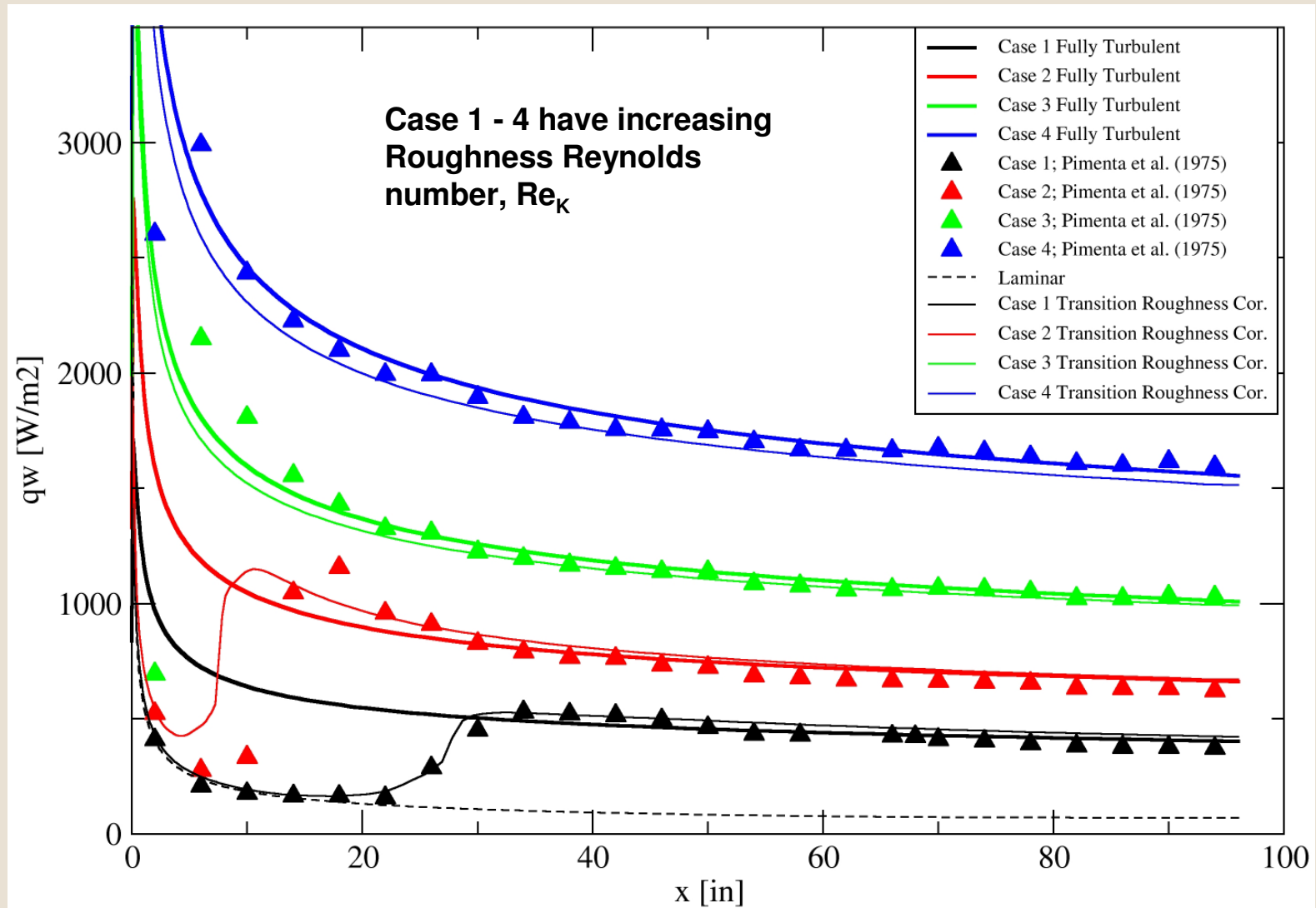


- Included Correlations for the effect of Roughness on transition onset

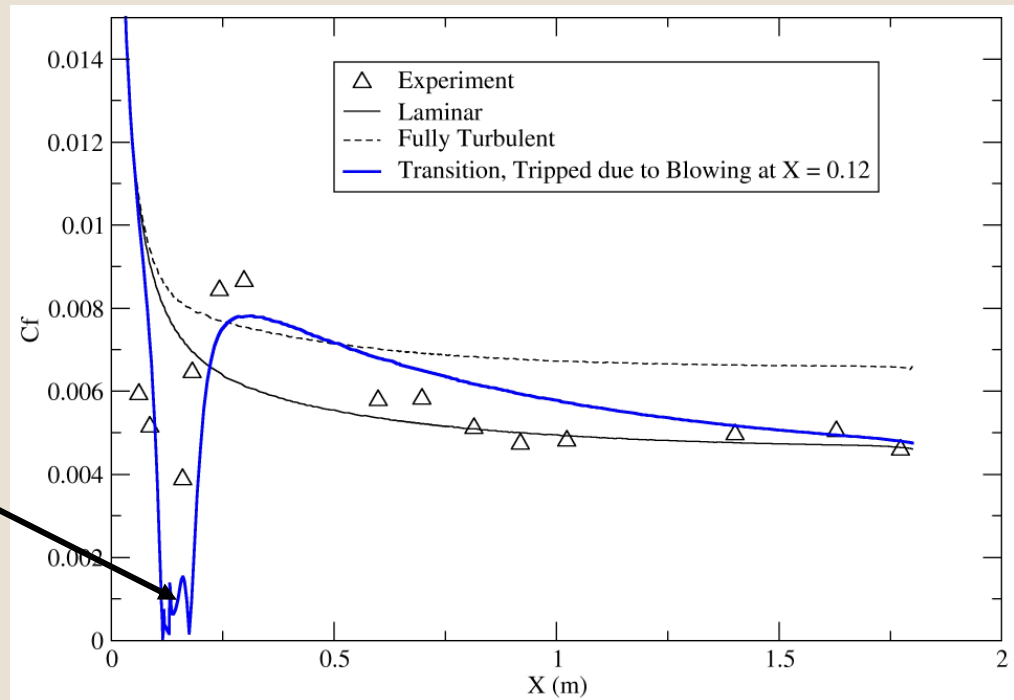




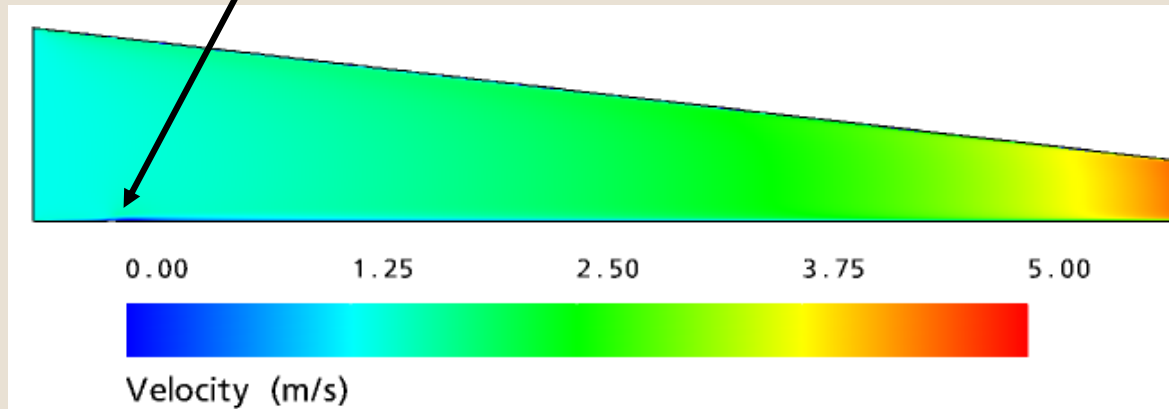
# Flat Plate Transition with Roughness



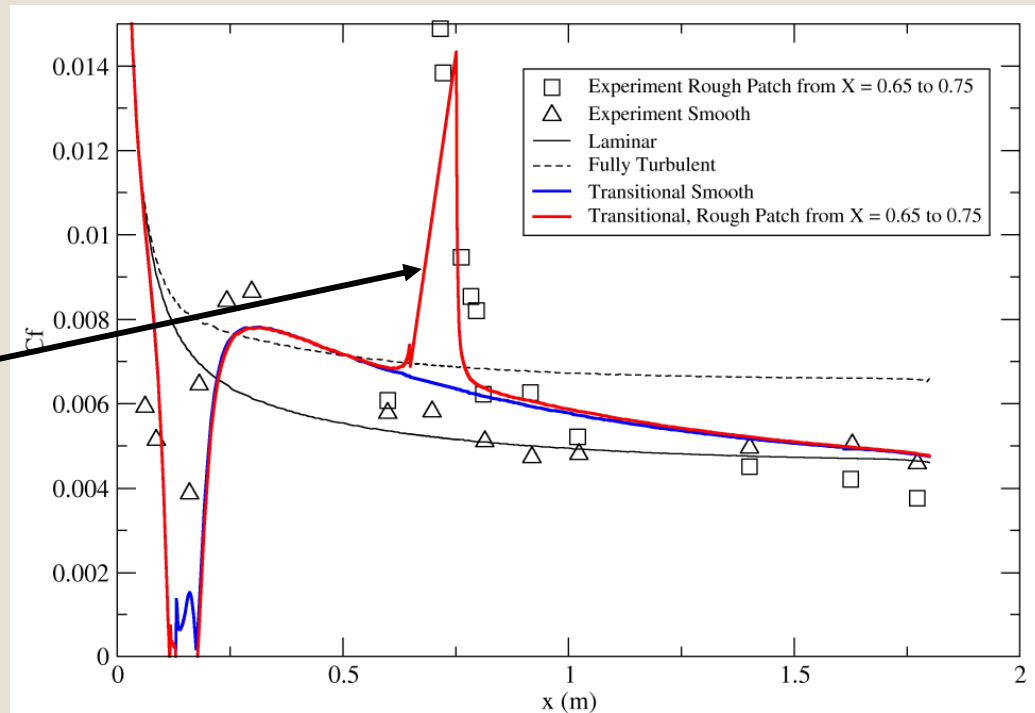
# Flat Plate Relaminarization



Trip due to Blowing



# Flat Plate Relaminarization with Roughness Strip



Roughness Strip

